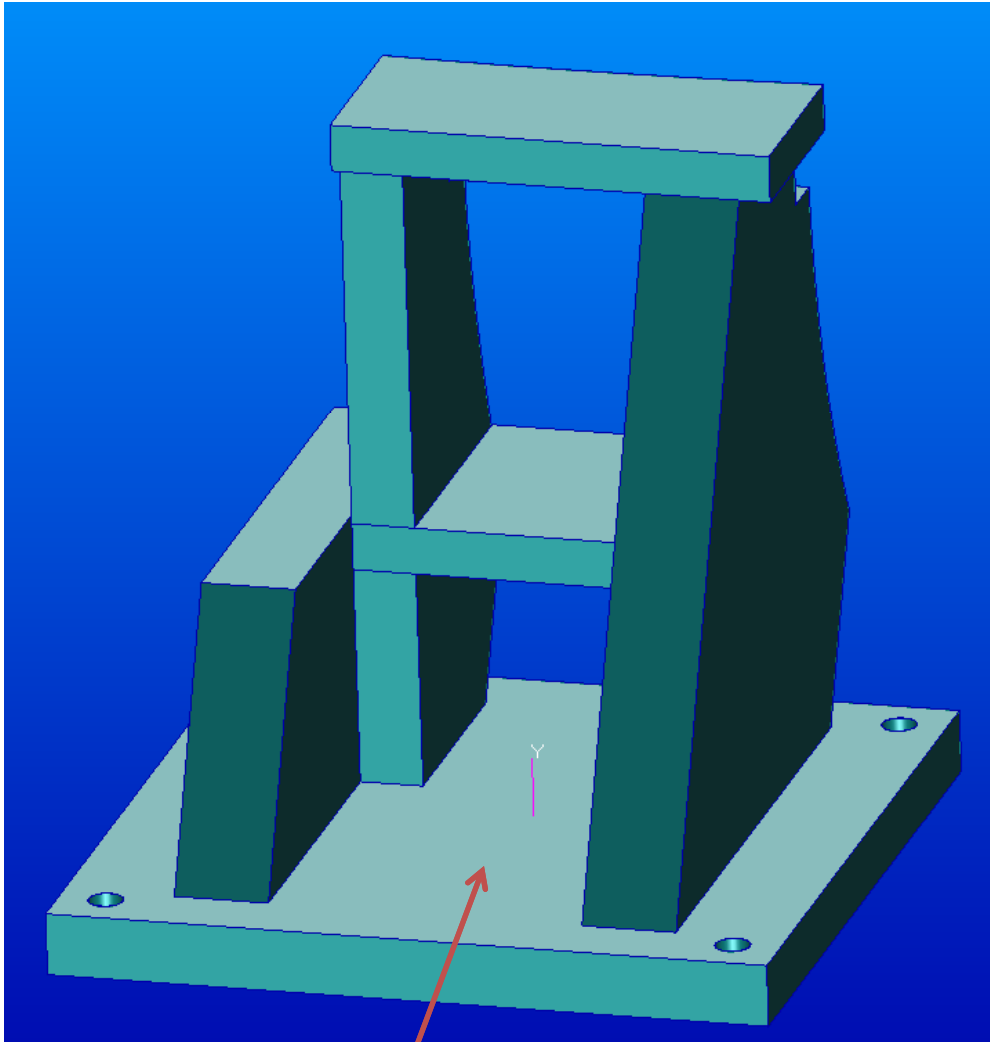


NSTXU PROJECT

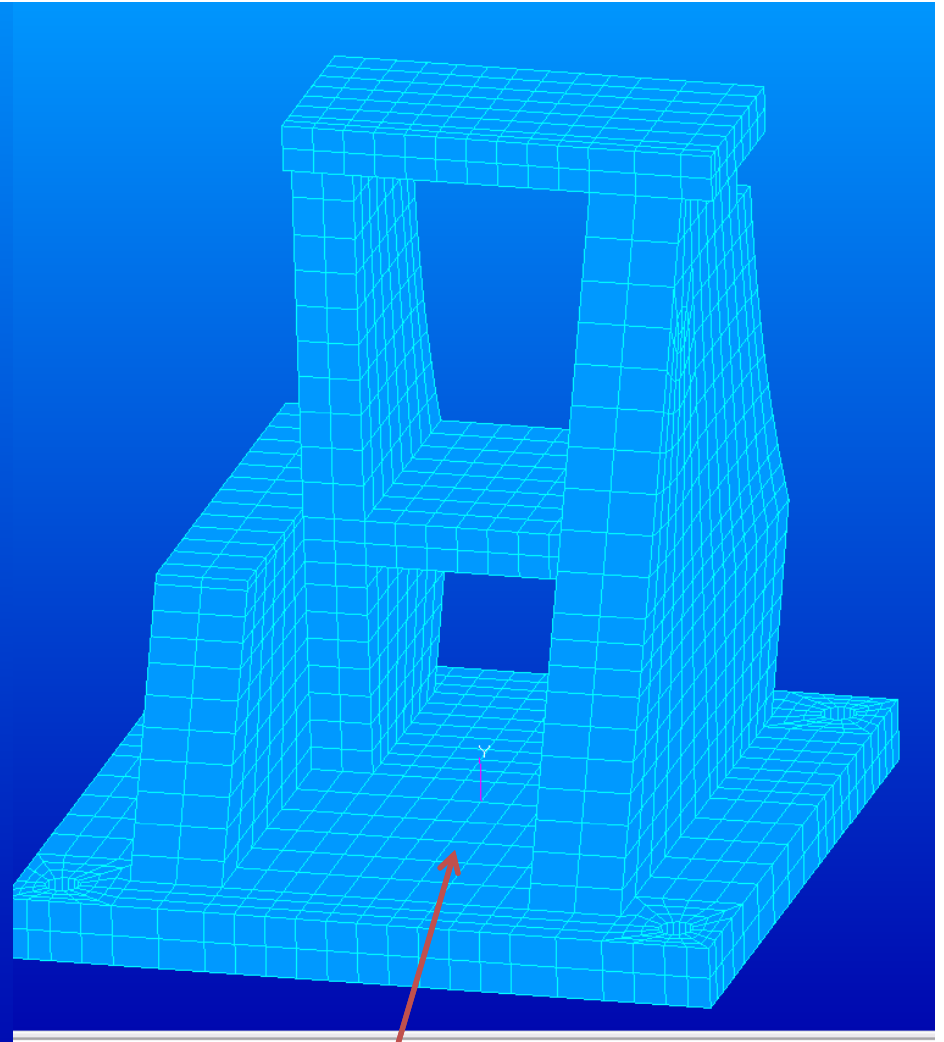
Vessel-Support-Chair.ppt

Complete simulation justification

P.R. March 2012



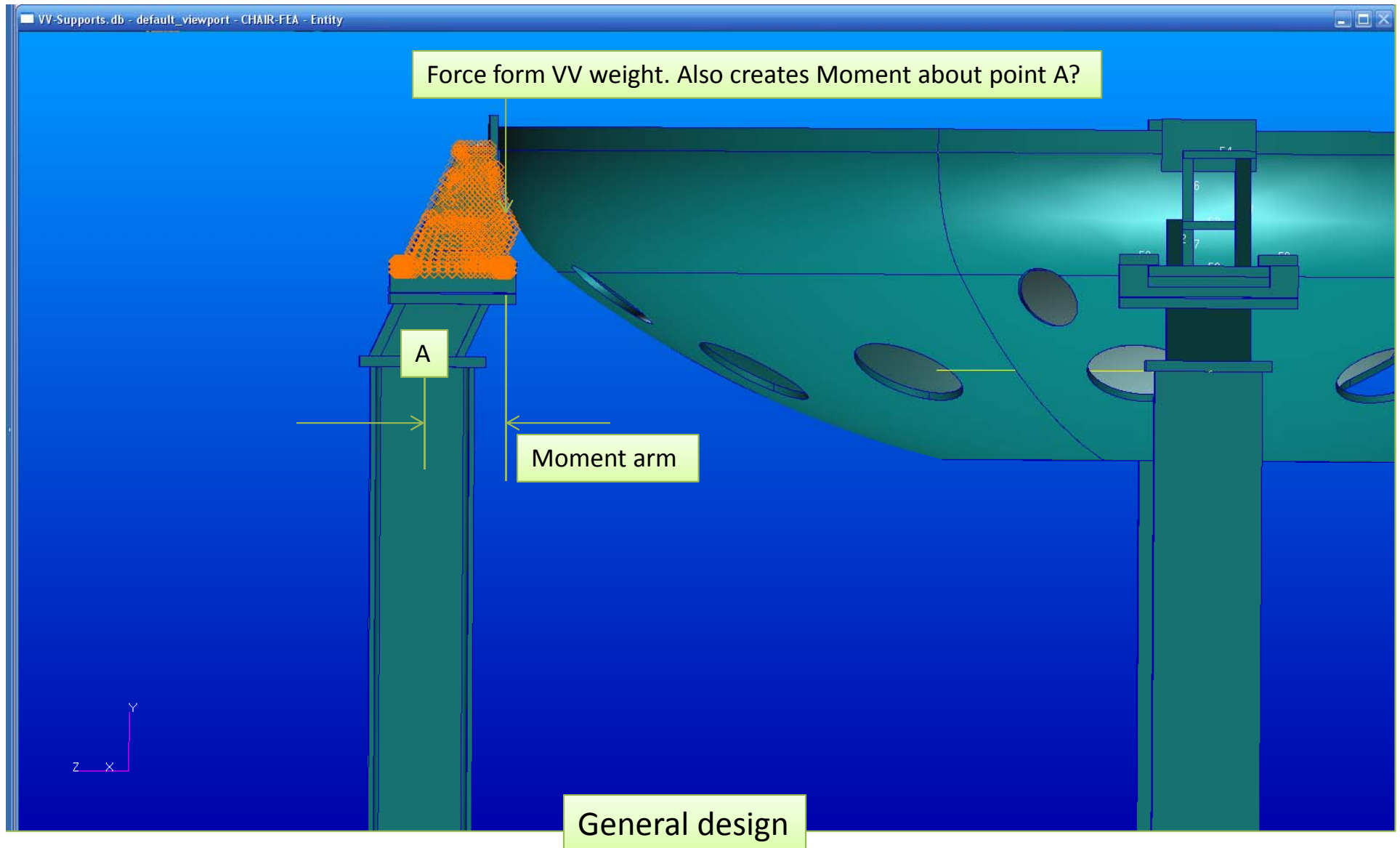
VV-support-attach-011812.igs



Nastran FEA all Hex8 elements

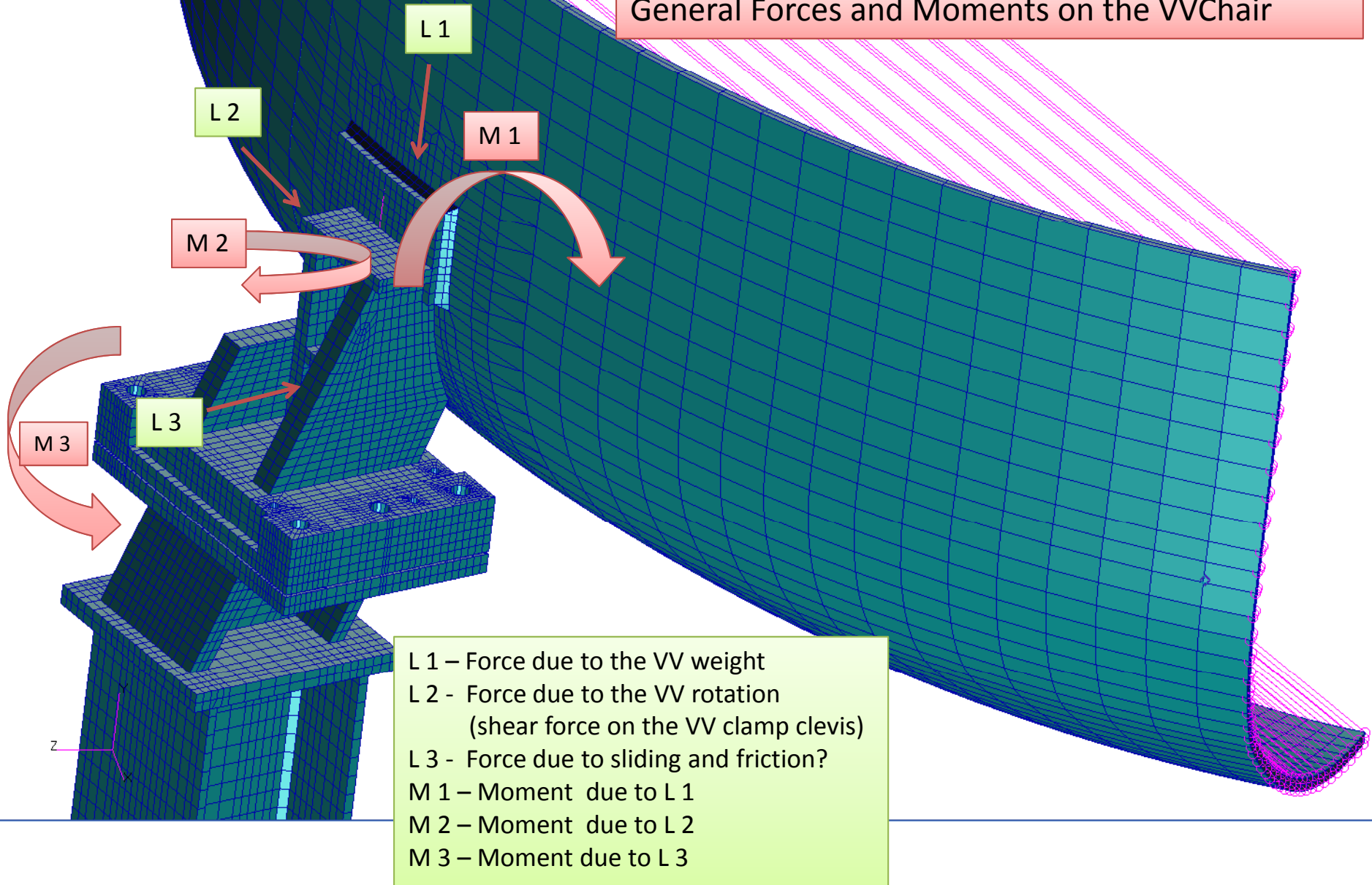
Question: How is this welded, assume weld size = $\frac{3}{8}$ inch?

Redesigned VV support chair



VV – Chair support configuration
[Complete geometry provided by M. Smith, 1/2012]

General Forces and Moments on the VVChair



Note: Force L 3 is a function of “L 1” and the coefficient of friction at the slide plate/cradle interface

```

$ NASTRAN input file created by the MSC FEA 2010.1.2 64-Bit input file
$ translator on March 08, 2012 at 12:06:33.
$ Nonlinear Static Analysis, Database
SOL 106
CEND
TITLE = MSC.NASTRAN JOB CREATED ON 17-FEB-12 AT 08:48:38
ECHO = NONE
MPC = 20134
SUBCASE 1
  TITLE=Non-Linear
  SUBTITLE=RUN5-Loads
  NLPARM = 1
  SPC = 2
  LOAD = 2
  DISPLACEMENT(SORT1,REAL)=ALL
  SPCFORCES(SORT1,REAL)=ALL
  STRAIN(SORT1,REAL,VONMISES,STRCUR,BILIN)=ALL
  STRESS(SORT1,REAL,VONMISES,BILIN)=ALL
$ Direct Text Input for this Subcase
BEGIN BULK
$ Direct Text Input for Bulk Data
PARAM POST 0
PARAM AUTOSPC YES
PARAM PRTMAXIM YES
NLPARM 1 10 AUTO 5 25 P NO
      .01

```

Complete model input file set-up

Note RUN #5 chosen for the demonstration (other runs are similar).

General Executive deck

```

$ Elements and Element Properties for region : sliding-Gaps
PGAP 2 0. 3.+7 500. 1.+7 .3 .3
-1.
$ Pset: "Sliding-Gaps" will be imported as: "pgap.2"
CGAP 94612 2 102192 127702 2
CGAP 94613 2 102195 128013 2

```

GAPs definition

MODEL SUMMARY

NUMBER OF GRID	POINTS	=	34815
NUMBER OF CGAP	ELEMENTS	=	660
NUMBER OF CHEXA	ELEMENTS	=	23184
NUMBER OF CPENTA	ELEMENTS	=	1336
NUMBER OF CTETRA	ELEMENTS	=	33
NUMBER OF RBE2	ELEMENTS	=	977

Total load due to the Pressure

Simulation grid and element statistics

Clevis loads (only for run #5)

```

RESULTS ABOUT ORIGIN OF SUPERELEMENT BASIC COORDINATE SYSTEM IN SUPERELEMENT BASIC SYSTEM COORDINATE
OLOAD RESULTANT
SUBCASE/ DAREA ID LOAD TYPE T1 T2 T3 R1 R2 R3
0 1 FX 3.576299E+04 ----- -3.998594E+04 -9.487278E+03 2.221206E+06 2.319899E+06 1.403519E+06
FY ----- -3.998594E+04 -9.487278E+03 3.723282E+05 1.632618E+05 -9.220052E+05
FZ ----- -9.487278E+03 3.723282E+05 1.632618E+05 -9.220052E+05
MX ----- 0.000000E+00 0.000000E+00 0.000000E+00
MY ----- 0.000000E+00 0.000000E+00 0.000000E+00
MZ ----- 0.000000E+00 0.000000E+00 0.000000E+00
TOTALS 3.576299E+04 -3.998594E+04 -9.487278E+03 2.593534E+06 2.483160E+06 4.815134E+05
1 MSC.NASTRAN JOB CREATED ON 17-FEB-12 AT 08:48:38 MARCH 8, 2012 MSC.NASTRAN

```

Resultant applied loads from the .f06 output

VV Support configuration FEA simulation

MPCs simulate
Cyclic Symmetry
3 – dof (1,2,3)
Hexa 8- elements

SPCs (1,2,3)
Fixity where
Braces are
located

Redesigned Chair

Slide plate

Cradle

RUN # 1

Static Test run #1 and Simulation general loads and fixities

Db --- Statics
File -- ChairVVStatics.bdf
This is a test run to check
Connections to the VV

40,000. lbs distributed over the VV thickness and 90 degree sector.
This value was taken from the previous analyses estimates.
It is applied as the distributed pressure.

SPCs in R-direction only

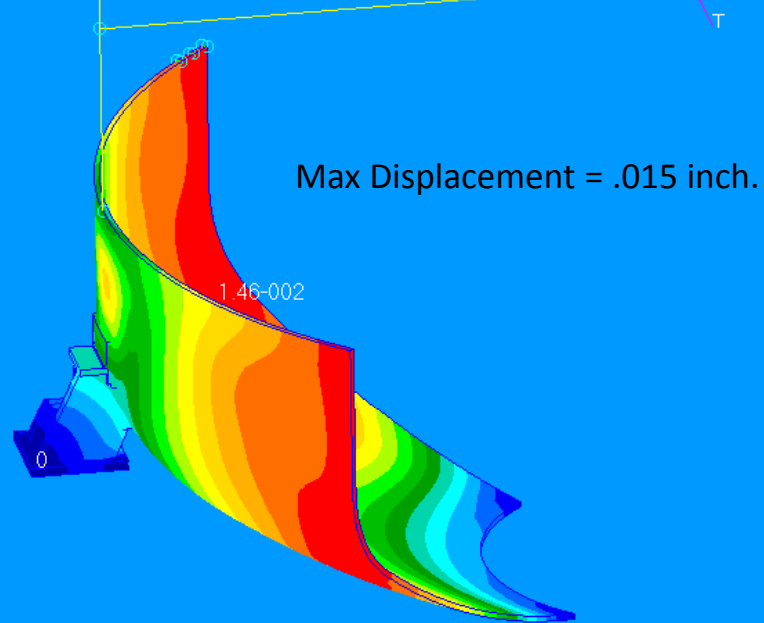
SPCs at four bolts
In 1,2,3 direction

Results are in ChairVVStatics.xdb

MSC FEA 2010.1.2 64-Bit 24-Feb-12 11:37:20

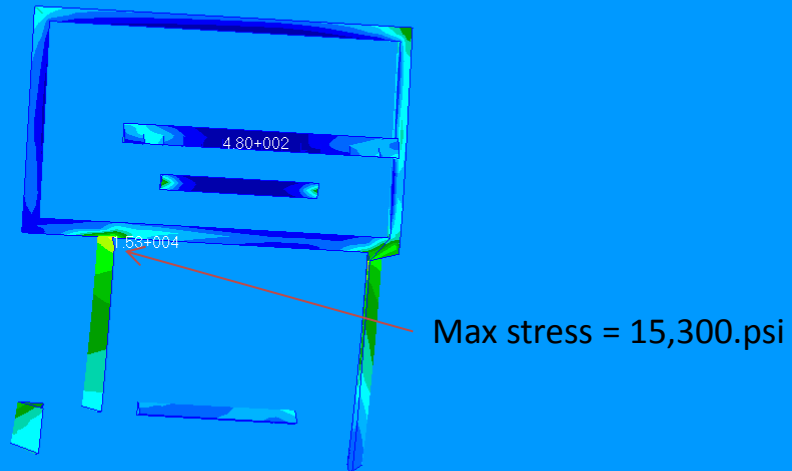
Fringe: Default, A1:Static Subcase, Displacements, Translational, Magnitude, (NON-LAYERED)

RUN # 1 Results



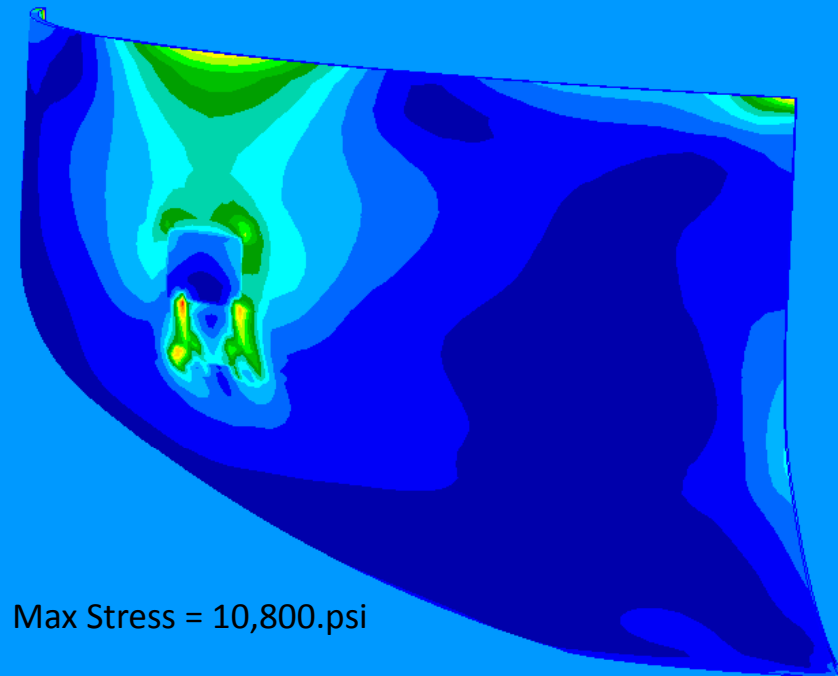
MSC FEA 2010.1.2 64-Bit 24-Feb-12 11:47:06

Fringe: Default, A1:Static Subcase, Stress Tensor, , von Mises, (NON-LAYERED)



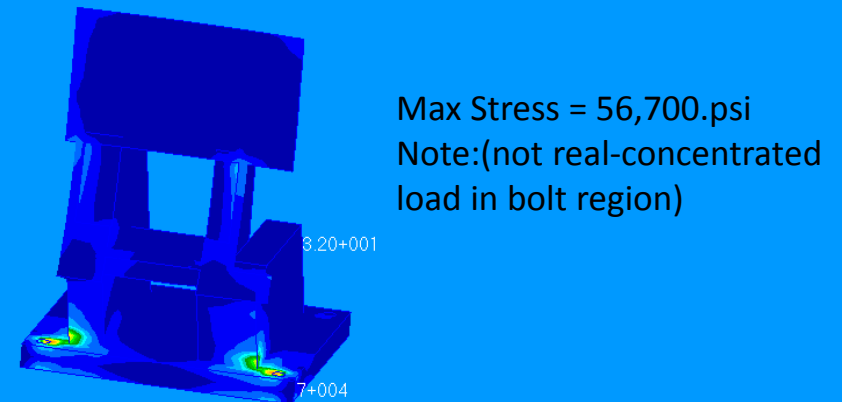
MSC FEA 2010.1.2 64-Bit 24-Feb-12 11:44:14

Fringe: Default, A1:Static Subcase, Stress Tensor, , von Mises, (NON-LAYERED)



MSC FEA 2010.1.2 64-Bit 24-Feb-12 11:52:31

Fringe: Default, A1:Static Subcase, Stress Tensor, , von Mises, (NON-LAYERED)

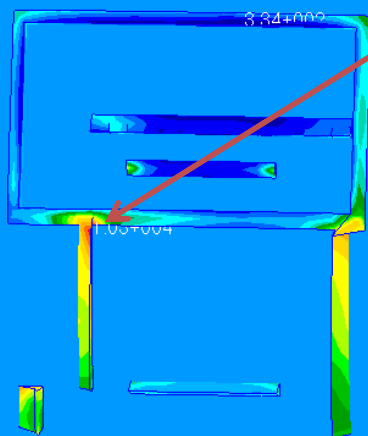
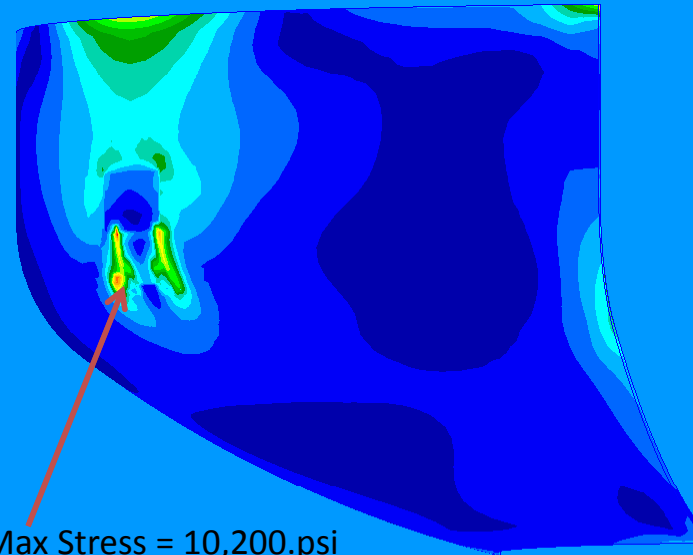
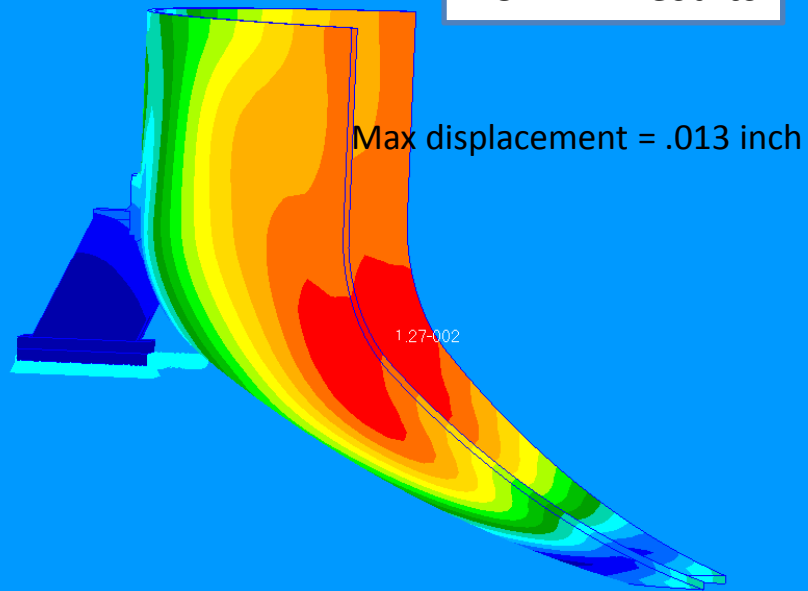


RUN #2

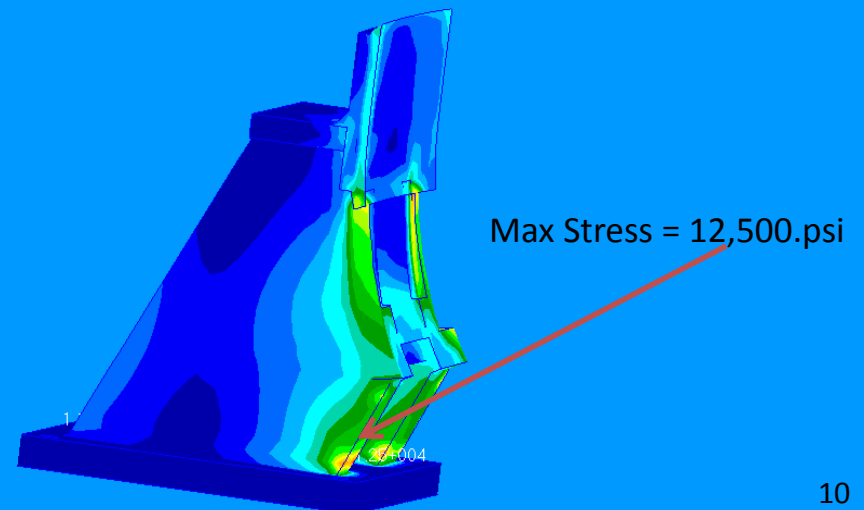


Added the slide plate and connected it via RBE2s (rigid links) to the chair with BC's as shown. Sliding in the R-direction nodes at the center of the slide plate. (Y in coord1=0.)

RUN #2 Results

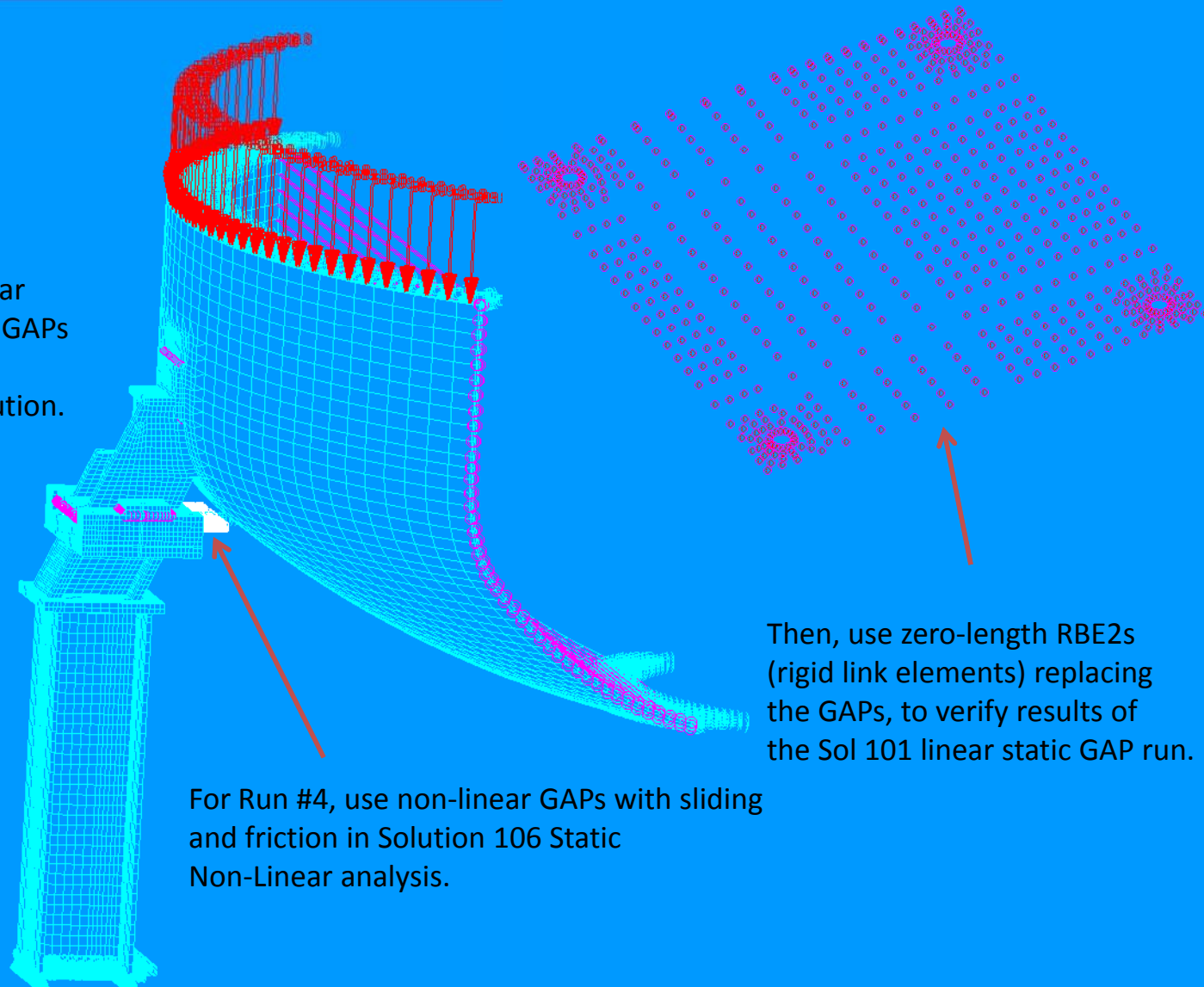


Welds between the Chair and the VV



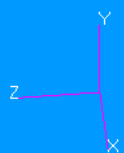
RUN #3

Sol 101 –First, set up a linear static run with zero-length GAPS along the pad surface that act as rigid links in this solution.

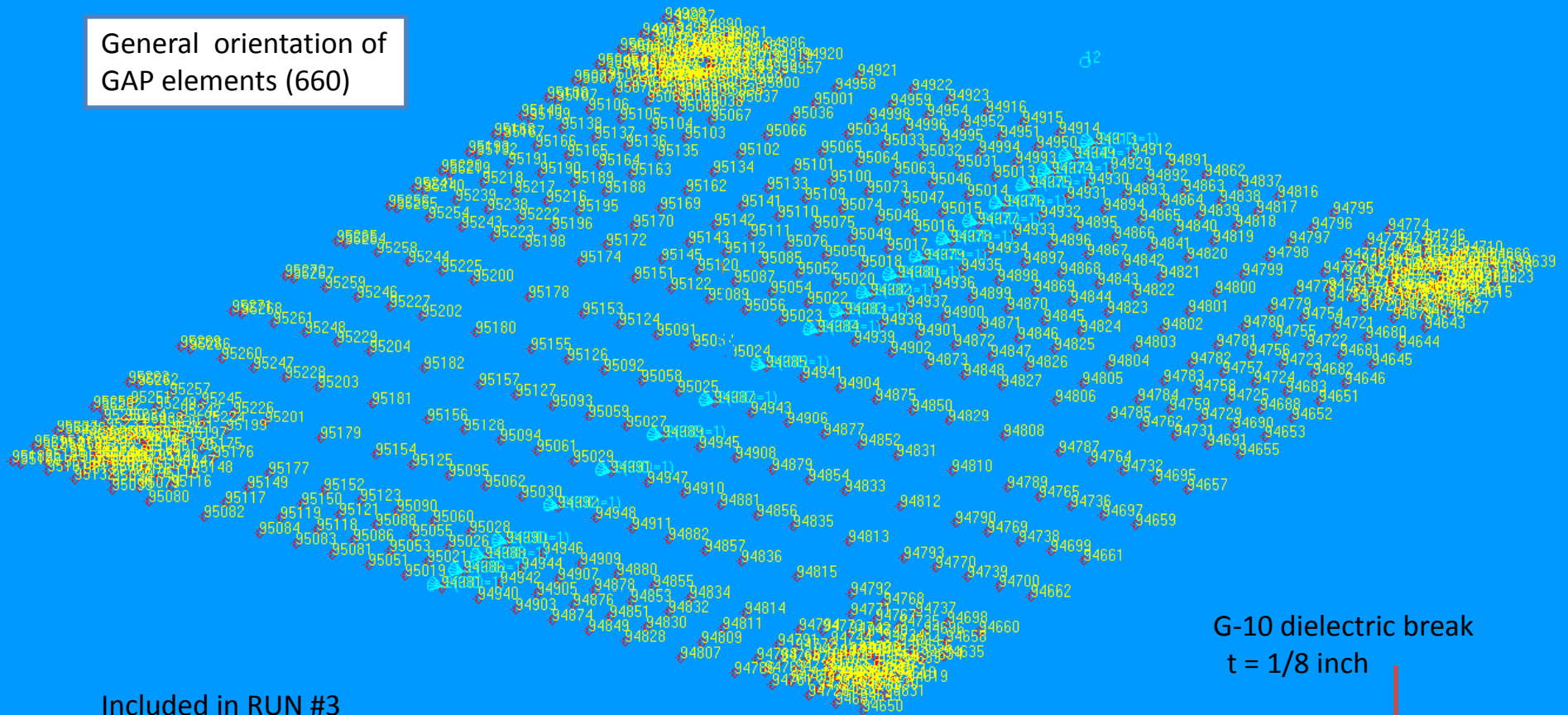


Then, use zero-length RBE2s (rigid link elements) replacing the GAPS, to verify results of the Sol 101 linear static GAP run.

For Run #4, use non-linear GAPS with sliding and friction in Solution 106 Static Non-Linear analysis.



General orientation of GAP elements (660)

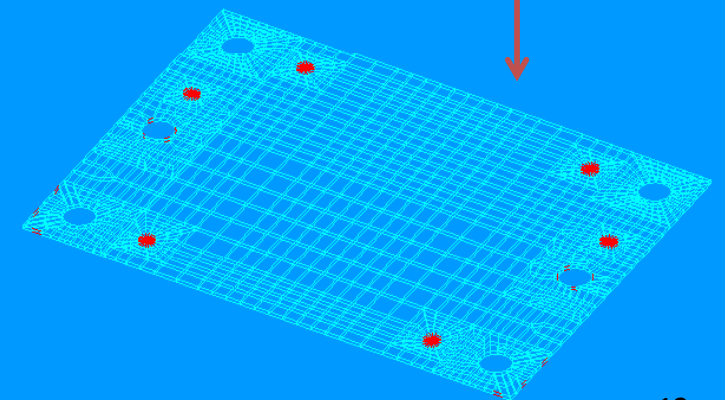


Included in RUN #3

```
PARAM      POST      0
PARAM      PRTMAXIM YES
$ Elements and Element Properties for region : sliding-Gaps
PGAP       1         0.         3.+7      500.      1.+7      .3      .3
          -1.
$ Pset: "sliding-Gaps" will be imported as: "pgap.1"
CGAP       94612     1         102192    127702    0.         -1.         0.
CGAP       94613     1         102195    128013    0.         -1.         0.
CGAP       94614     1         102199    128016    0.         -1.         0.
CGAP       94615     1         102201    128015    0.         -1.         0.
CGAP       94616     1         102204    131457    0.         -1.         0.
```

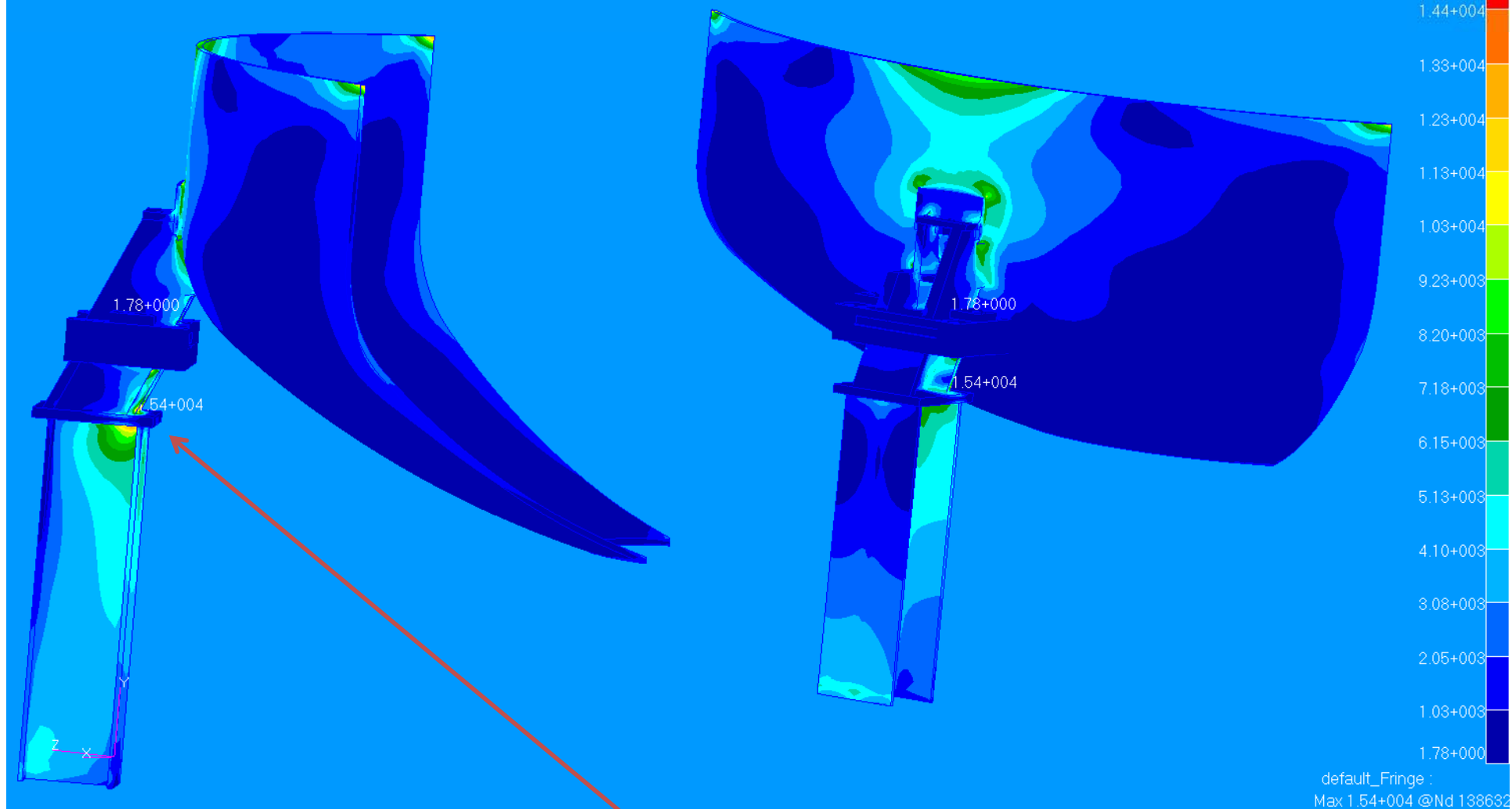


G-10 dielectric break
t = 1/8 inch



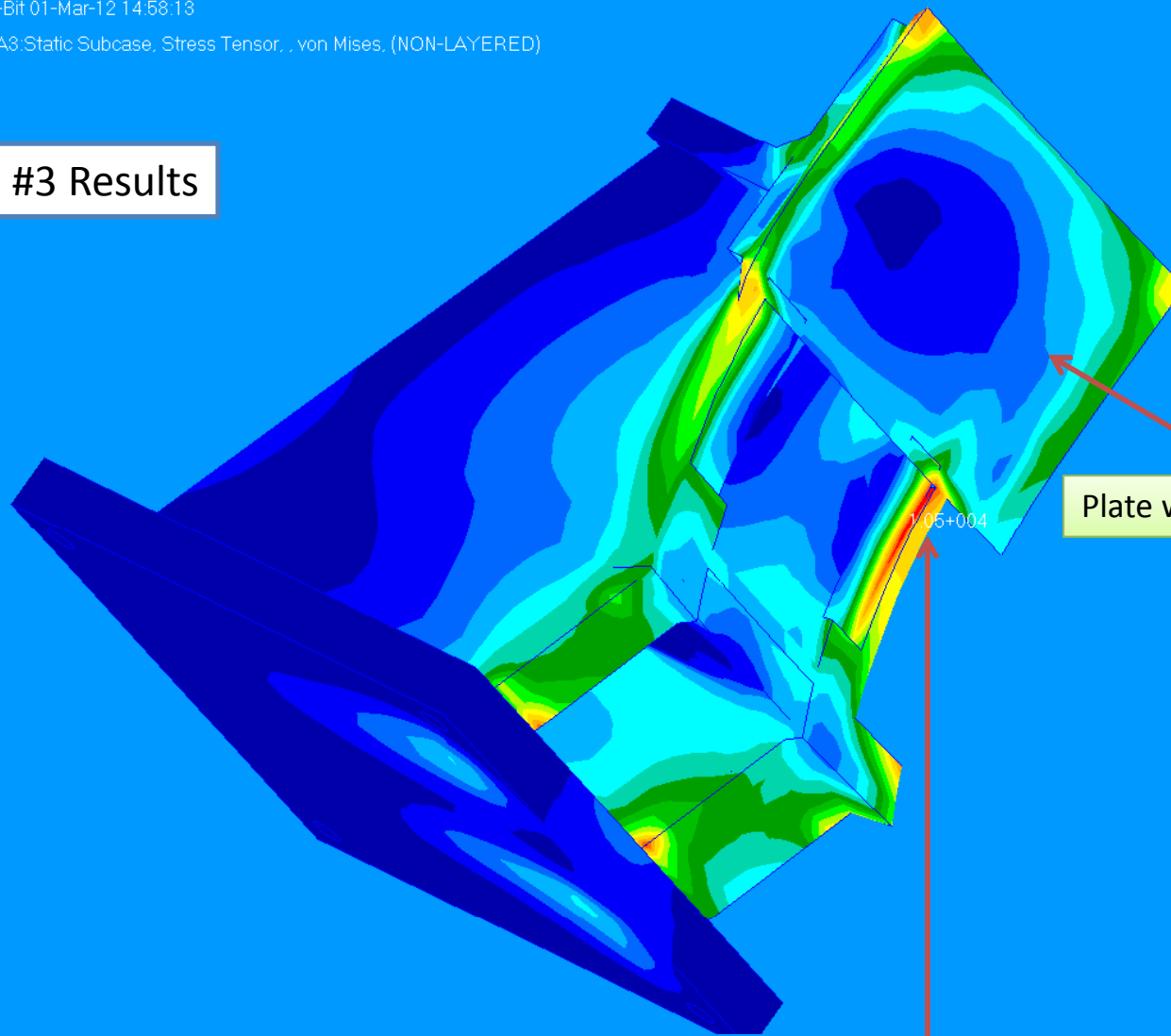
Between the cradle and Vertical support

RUN #3 Results



Maximum Von Mises stress on the total system for this run = 15,400. psi.

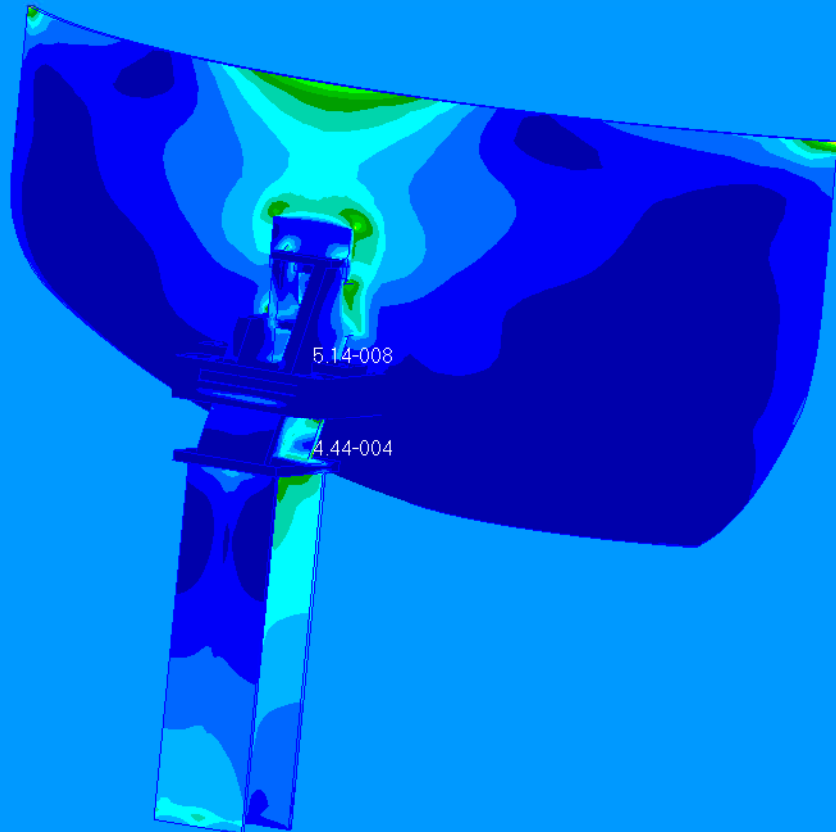
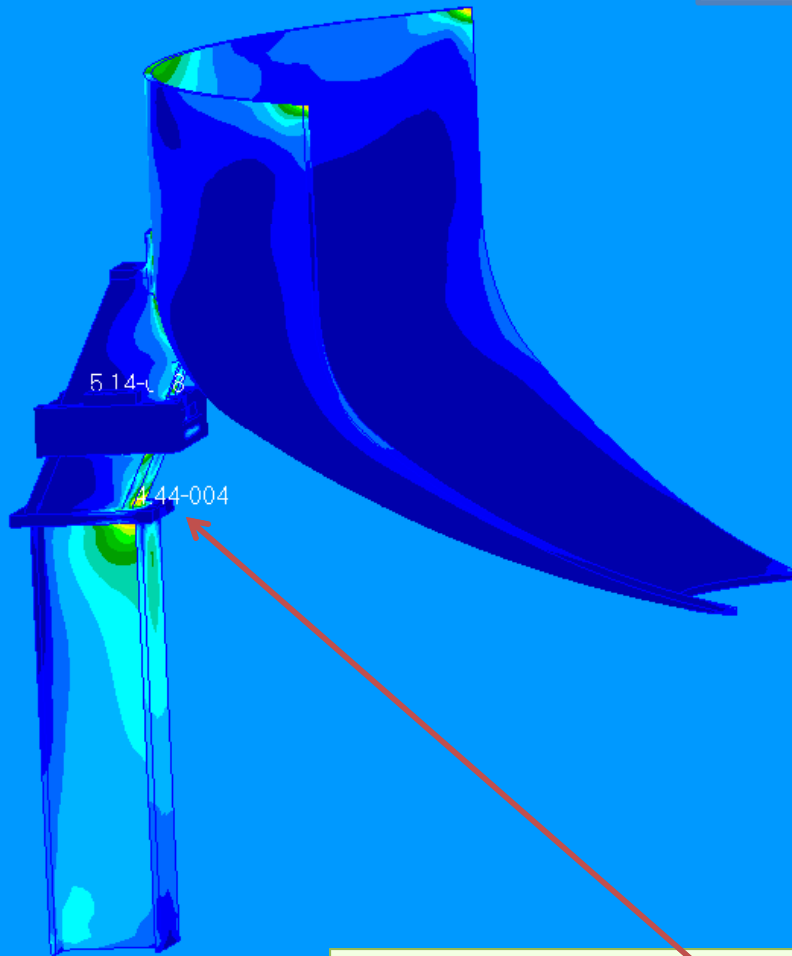
RUN #3 Results



default_Fringe :
Max 1.05+004 @Nd 13113

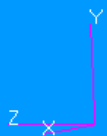
VV Chair with welds, Maximum Von Mises Stress = 10,500.psi

RUN #3 Results



Maximum strain on the total system for this run = .00044

default_Fringe :
Max 4.44-004 @Nd 138632



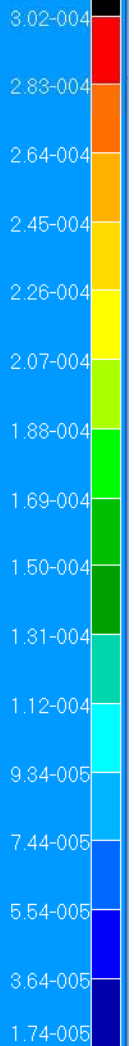
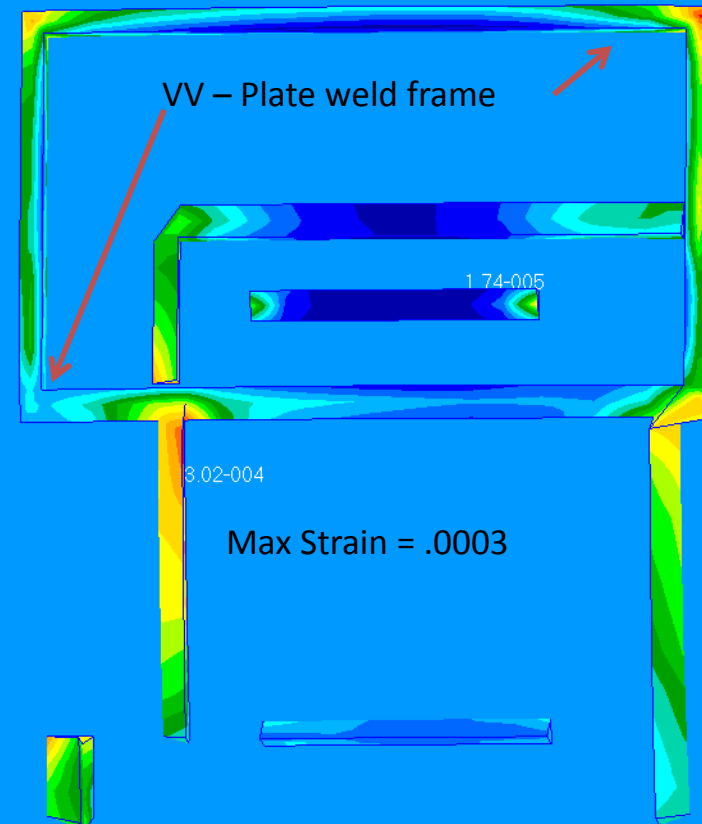
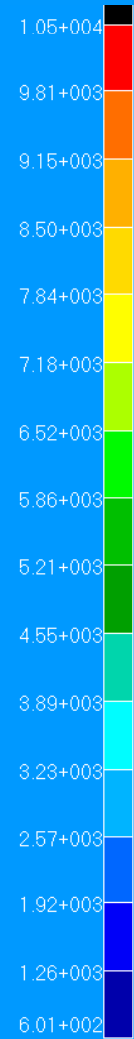
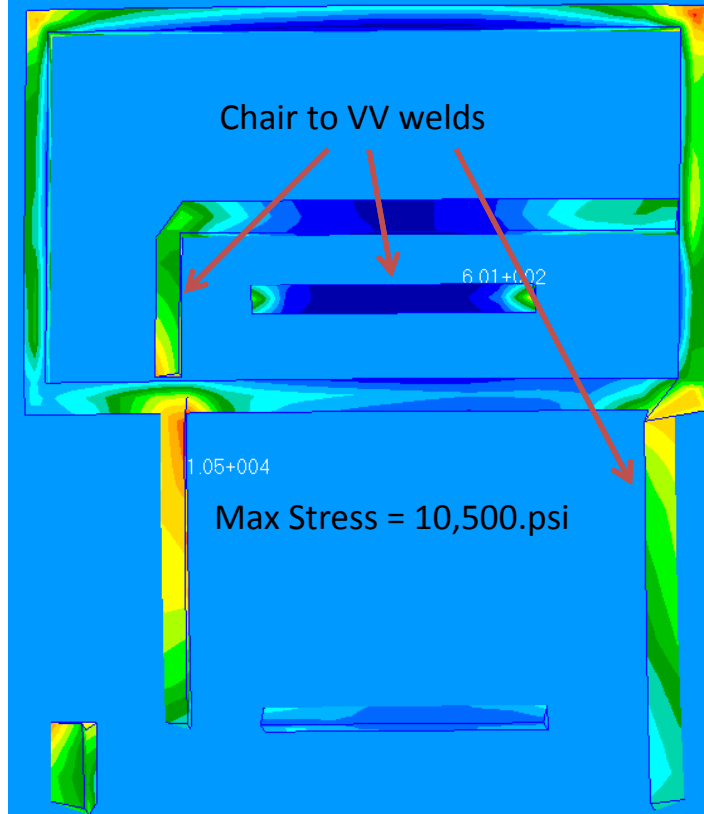
MSC FEA 2010.1.2 64-Bit 02-Mar-12 08:19:23

Fringe: RUN3_Loads, A3:Static Subcase, Strain Tensor, ... (NON-LAYERED)

MSC FEA 2010.1.2 64-Bit 02-Mar-12 08:24:09

Fringe: RUN3_Loads, A3:Static Subcase, Stress Tensor, von Mises, (NON-LAYERED)

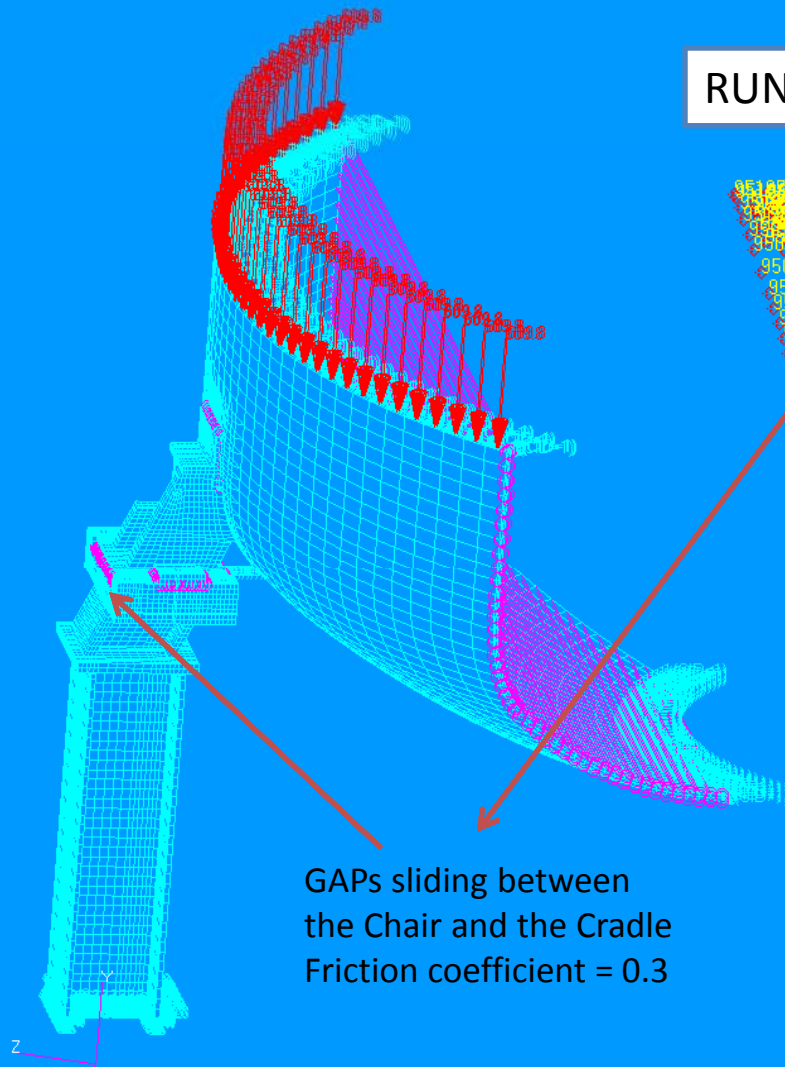
RUN #3 Results



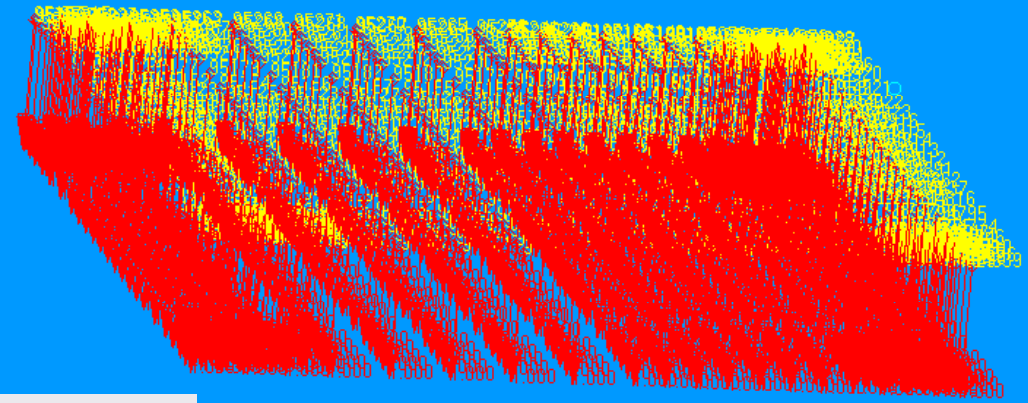
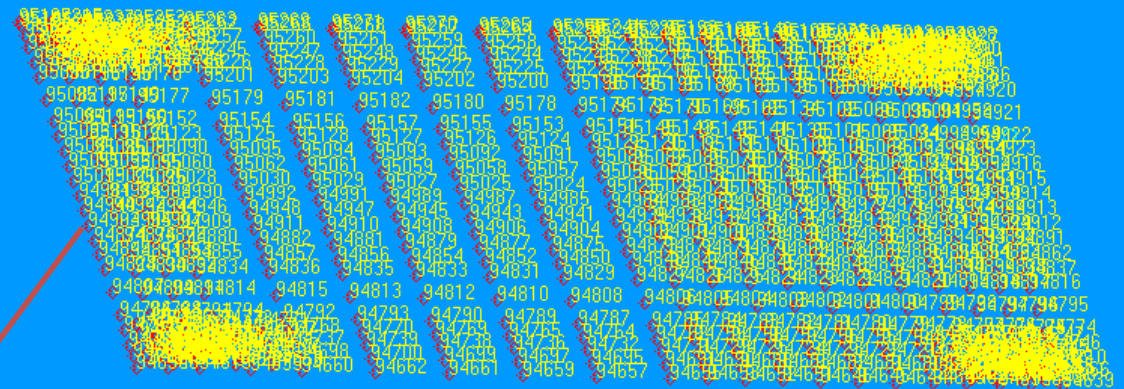
default_Fringe :
Max 3.02-004 @Nd 13113
Min 1.74-005 @Nd 12001

Results from the pure linear run where GAPS are replaced by the RBE2s

RUN #4 Non Linear Static Solution (106)



GAPs sliding between
the Chair and the Cradle
Friction coefficient = 0.3

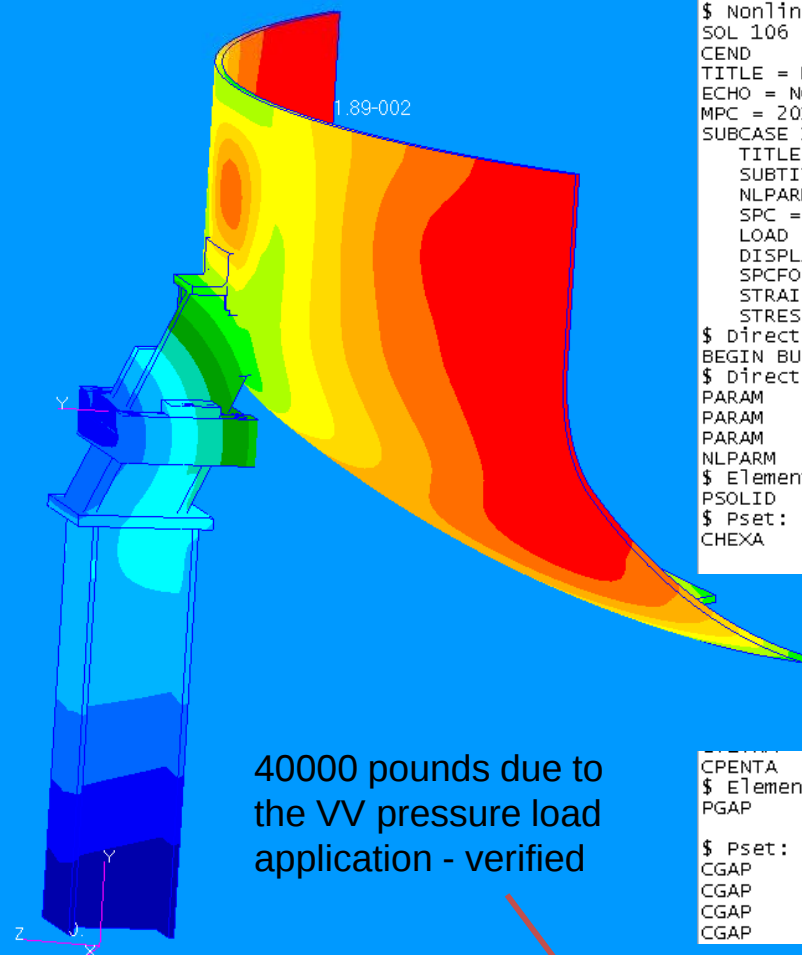


GAPs orientation in the -Y direction

Load Case "RUN4-Loads"

	Type	Scale Factor	Priority
Slide_Fic_cen	Displacement	1.	Add
Support_Bot	Displacement	1.	Add
VV-Fixity	Displacement	1.	Add
VV-Fixity-Bot	Displacement	1.	Add
Pres40000	Pressure	1.	Add

RUN #4 -Results



40000 pounds due to the VV pressure load application - verified

```

$ Nonlinear Static Analysis, Database
SOL 106
CEND
TITLE = MSC.NASTRAN JOB CREATED ON 17-FEB-12 AT 08:48:38
ECHO = NONE
MPC = 20134
SUBCASE 1
  TITLE=Non Linear
  SUBTITLE=RUN4-Loads
  NLPARM = 1
  SPC = 2
  LOAD = 2
  DISPLACEMENT(SORT1,REAL)=ALL
  SPCFORCES(SORT1,REAL)=ALL
  STRAIN(SORT1,REAL,VONMISES,STRUCUR,BILIN)=ALL
  STRESS(SORT1,REAL,VONMISES,BILIN)=ALL
$ Direct Text Input for this Subcase
BEGIN BULK
$ Direct Text Input for Bulk Data
PARAM POST 0
PARAM AUTOSPC YES
PARAM PRTMAXIM YES
NLPARM 1 10 AUTO 5 25 NO
$ Elements and Element Properties for region : HexwedTet
PSOLID 1 1 0
$ Pset: "HexwedTet" will be imported as: "psolid.1"
CHEXA 2793 1 14407 14420 14419 14406 14606 14627
      14626 14605
  
```

```

CPENTA 95277 1 16149 12736 12733 16152 12576 12573
$ Elements and Element Properties for region : Sliding-Gaps
PGAP 2 0. 3.+7 500. 1.+7 .3 .3
      -1.
$ Pset: "Sliding-Gaps" will be imported as: "pgap.2"
CGAP 94612 2 102192 127702 2
CGAP 94613 2 102195 128013 2
CGAP 94614 2 102199 128016 2
CGAP 94615 2 102201 128015 2
  
```

RESULTANTS ABOUT ORIGIN OF SUPERELEMENT BASIC COORDINATE SYSTEM IN SUPERELEMENT BASIC SYSTEM COORDINATES.

SUBCASE/ DAREA ID	LOAD TYPE	T1	T2	T3	R1	R2	R3
0	FX	0.000000E+00	----	----	----	0.000000E+00	0.000000E+00
0	FY	----	-3.998594E+04	----	2.221206E+06	----	-9.220052E+05
	FZ	----	----	0.000000E+00	0.000000E+00	0.000000E+00	----
	MX	----	----	----	0.000000E+00	----	----
	MY	----	----	----	----	0.000000E+00	----
	MZ	----	----	----	----	----	0.000000E+00
TOTALS		0.000000E+00	-3.998594E+04	0.000000E+00	2.221206E+06	0.000000E+00	-9.220052E+05

MSC.NASTRAN JOB CREATED ON 17-FEB-12 AT 08:48:38

default_Fringe :
Max 1.89-002 @Nd 5903

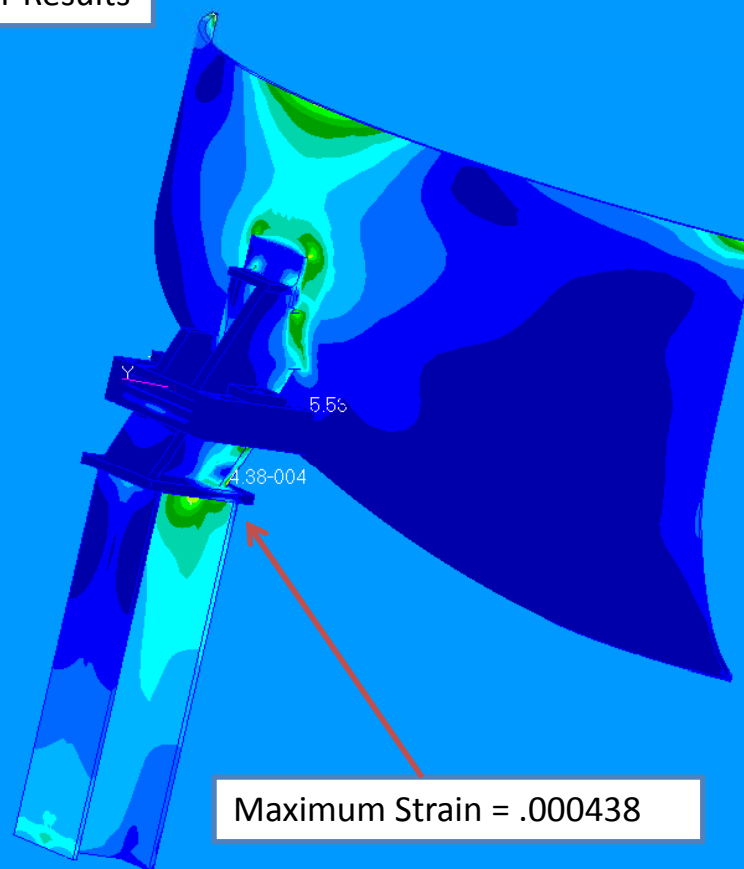
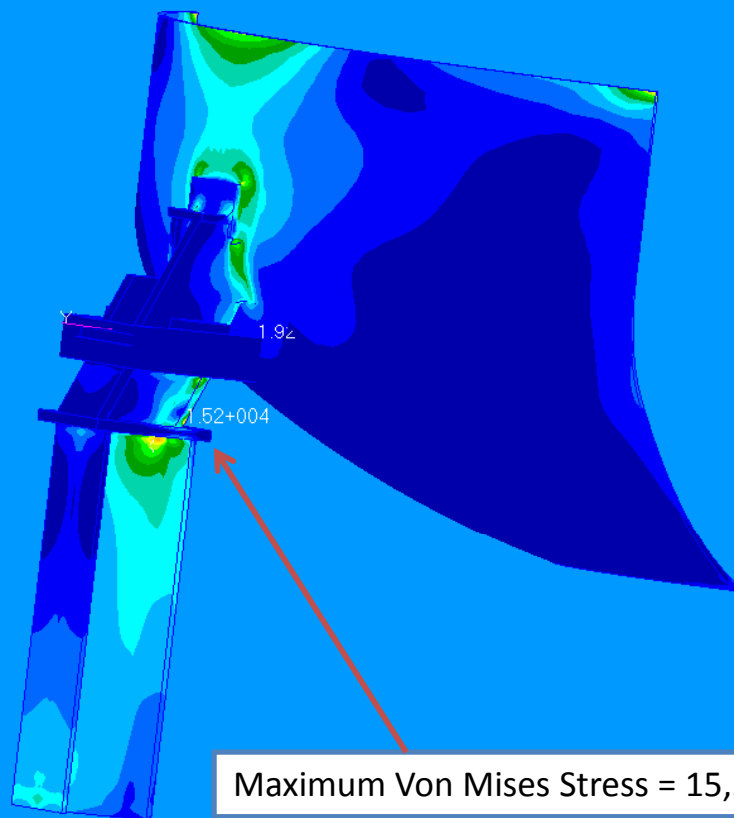
MSC FEA 2010.1.2 64-Bit 06-Mar-12 13:42:35

Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Strain Tensor, ... (NON-LAYERED)

MSC FEA 2010.1.2 64-Bit 06-Mar-12 14:00:43

Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Stress Tensor, , von Mises, (NON-LAYERED)

RUN #4-Results



default_Fringe :
Max 4.38-004 @Nd 13863

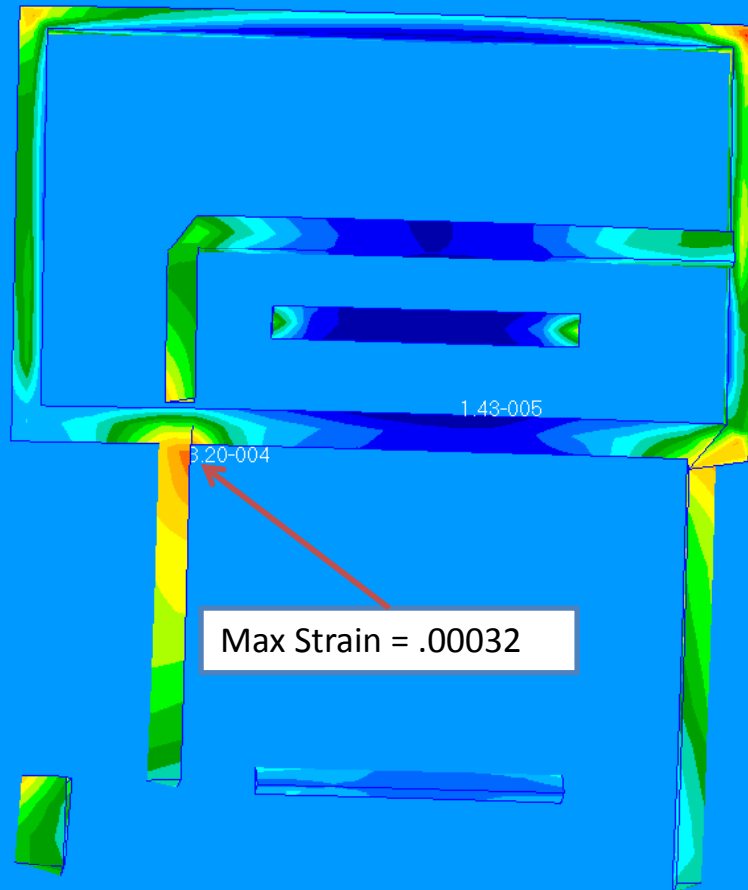
Note: These Non Linear results are very close to those calculated in linear RUN #3. This is actually expected because non-linear GAP sliding is small (See following slides). This confirms that a linear analysis can be used as effectively as a non-linear analysis for solving this calculation.

MSC FEA 2010.1.2 64-Bit 06-Mar-12 14:22:22

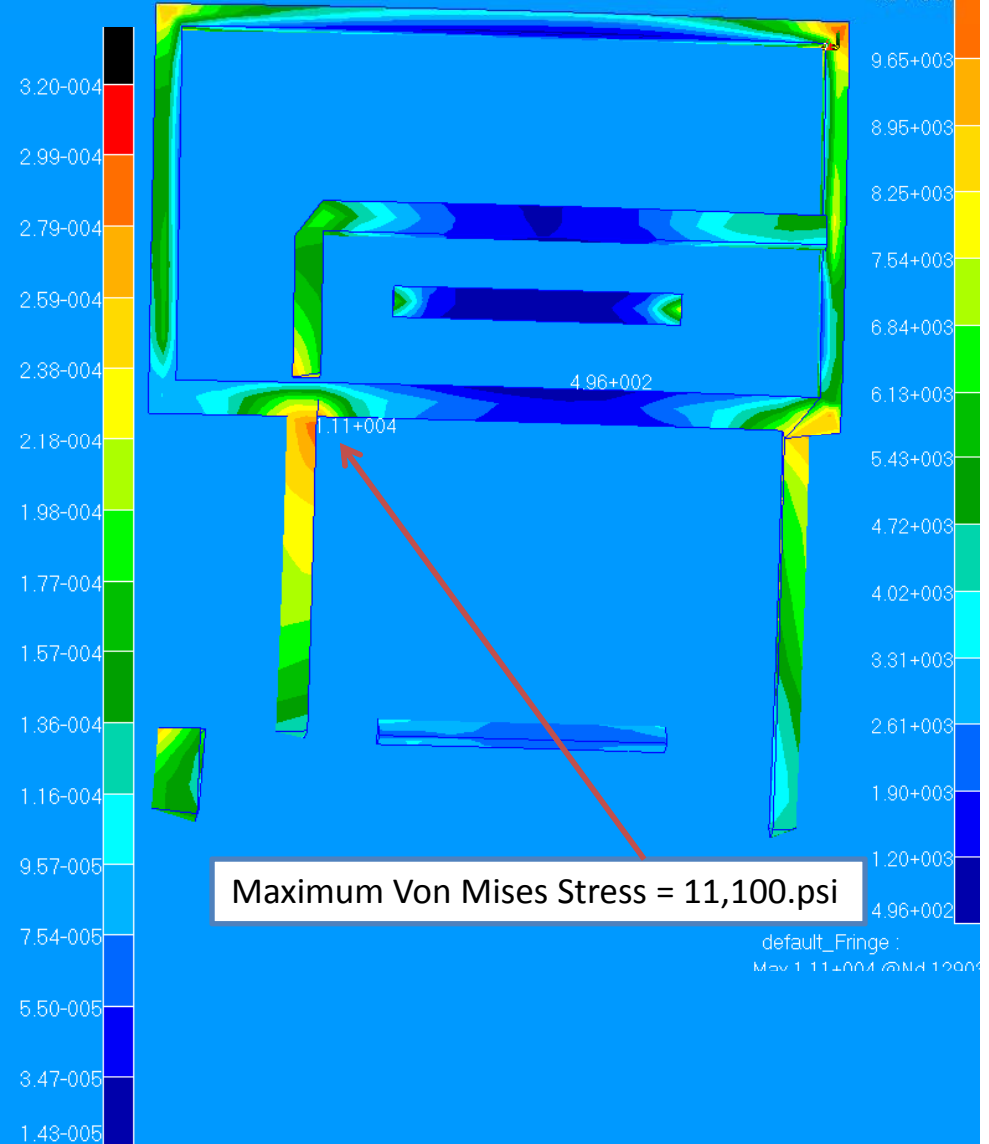
Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Stress Tensor, , von Mises, (NON-LAYERED)

MSC FEA 2010.1.2 64-Bit 06-Mar-12 14:34:18

Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Strain Tensor, , (NON-LAYERED)



RUN #4-Results

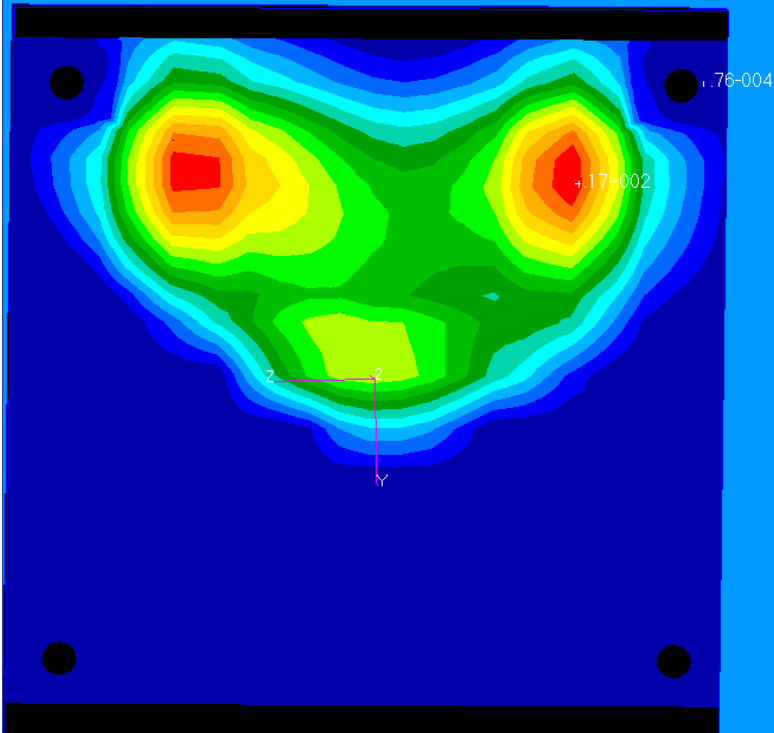


default_Fringe :
Max 1.11+004 @Nd 12004

Please see Run #3 (Slide #16) results for comparison

RUN #4 – Load & Slip Results for Chair Slide Plate

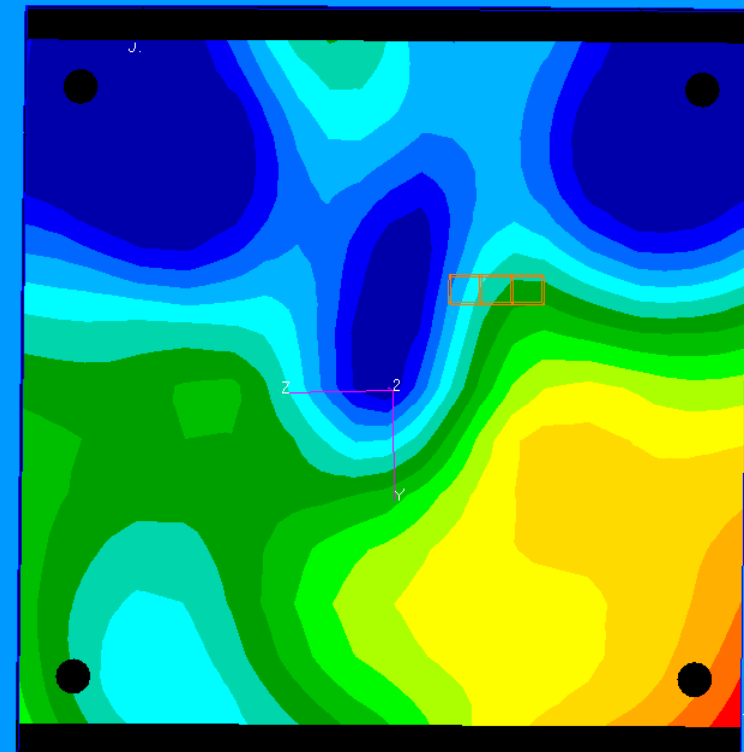
Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Gap Results, Force, Magnitude, At Center



Gaps Load transfer distribution

MSC FEA 2010.1.2 64-Bit 06-Mar-12 15:19:23

Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Gap Results, Slip, Magnitud



default_Fringe :
Max 1.42e-04 @ND 131741

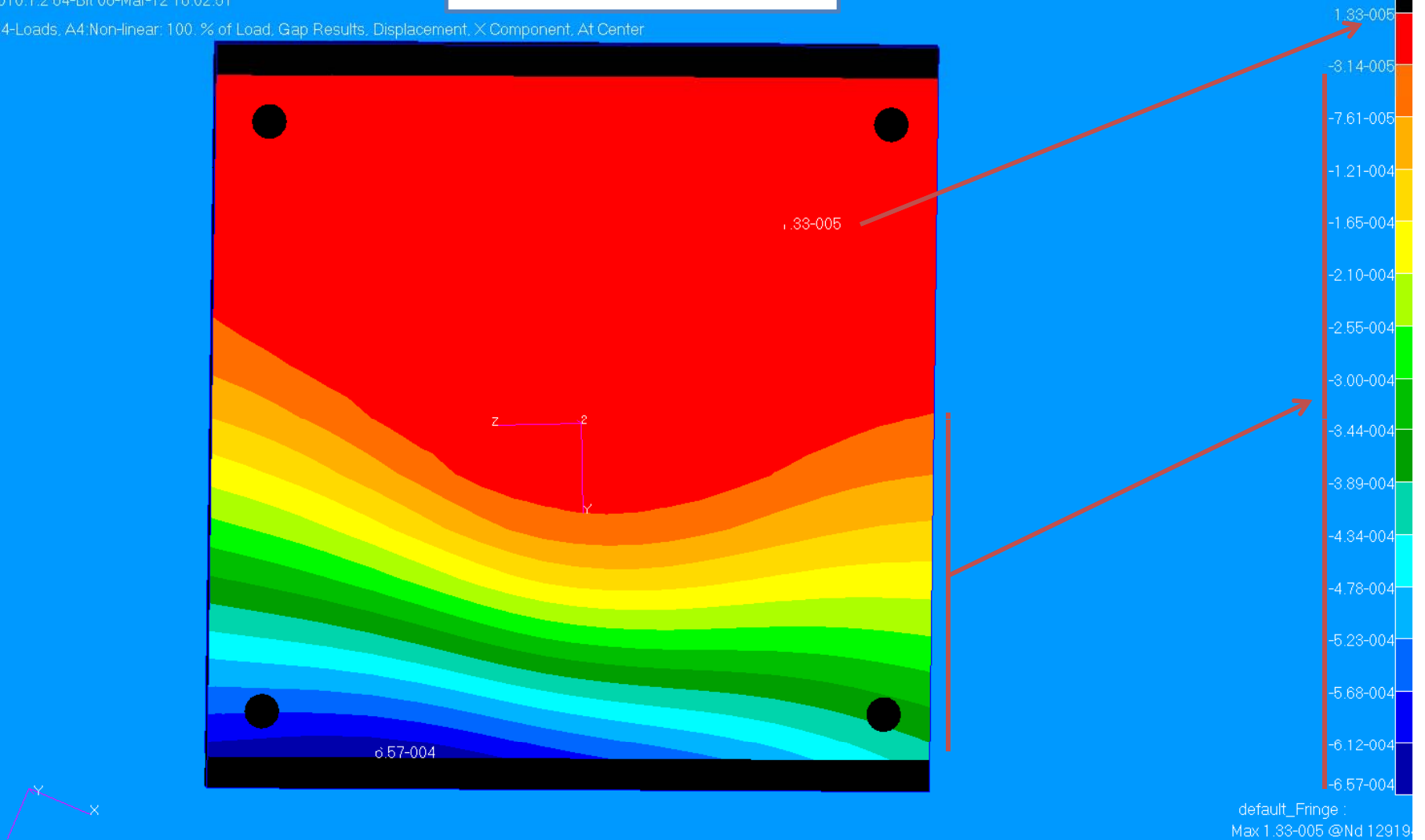
Gaps slipping distribution

Note : These results are based on the coefficient of friction = 0.3 for GAP "Y" and "Z" transverse directions

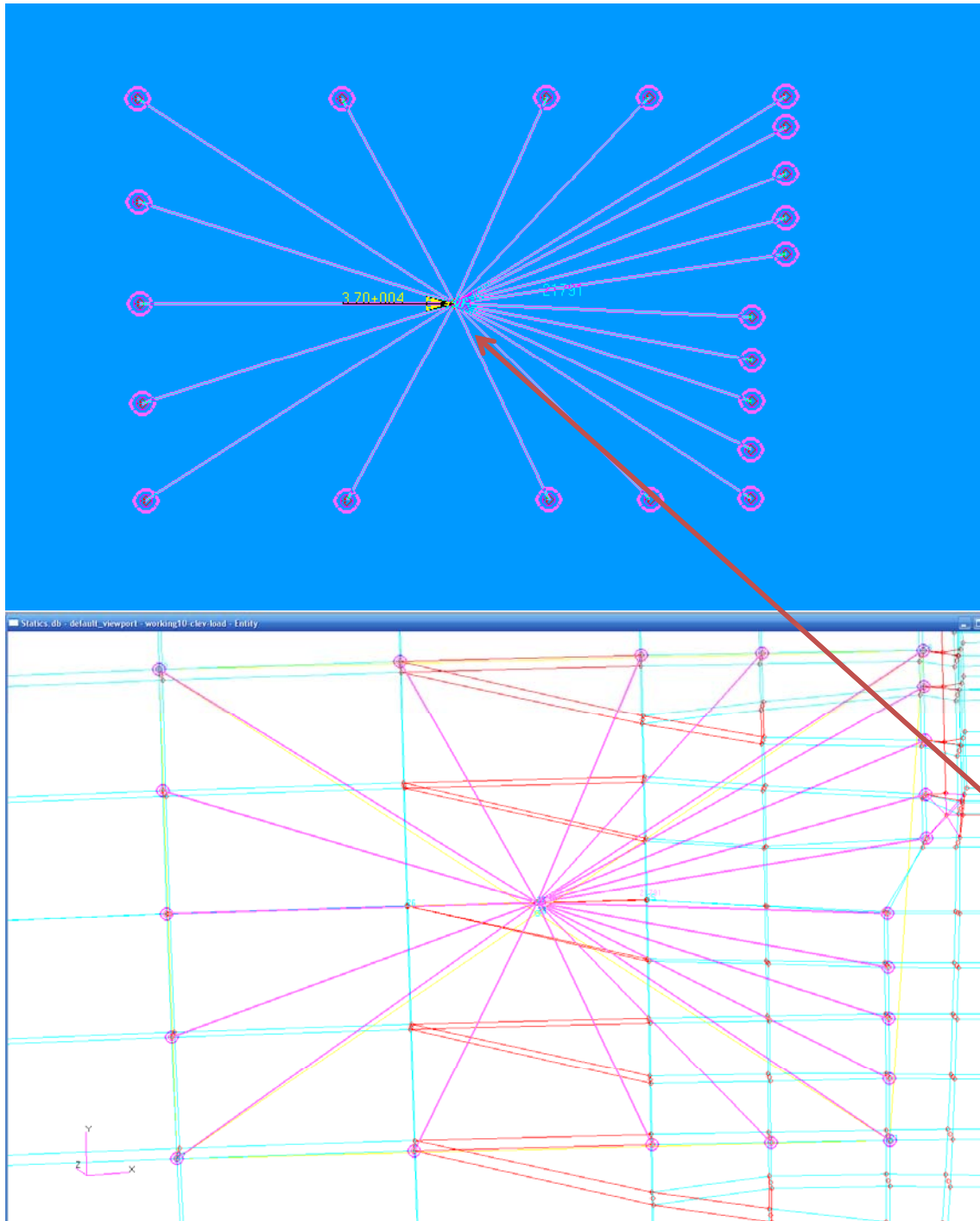
RUN #4 – GAP Contact Results

MSC FEA 2010.1.2 64-Bit 06-Mar-12 16:02:51

Fringe: RUN4-Loads, A4:Non-linear: 100. % of Load, Gap Results, Displacement, X Component, At Center



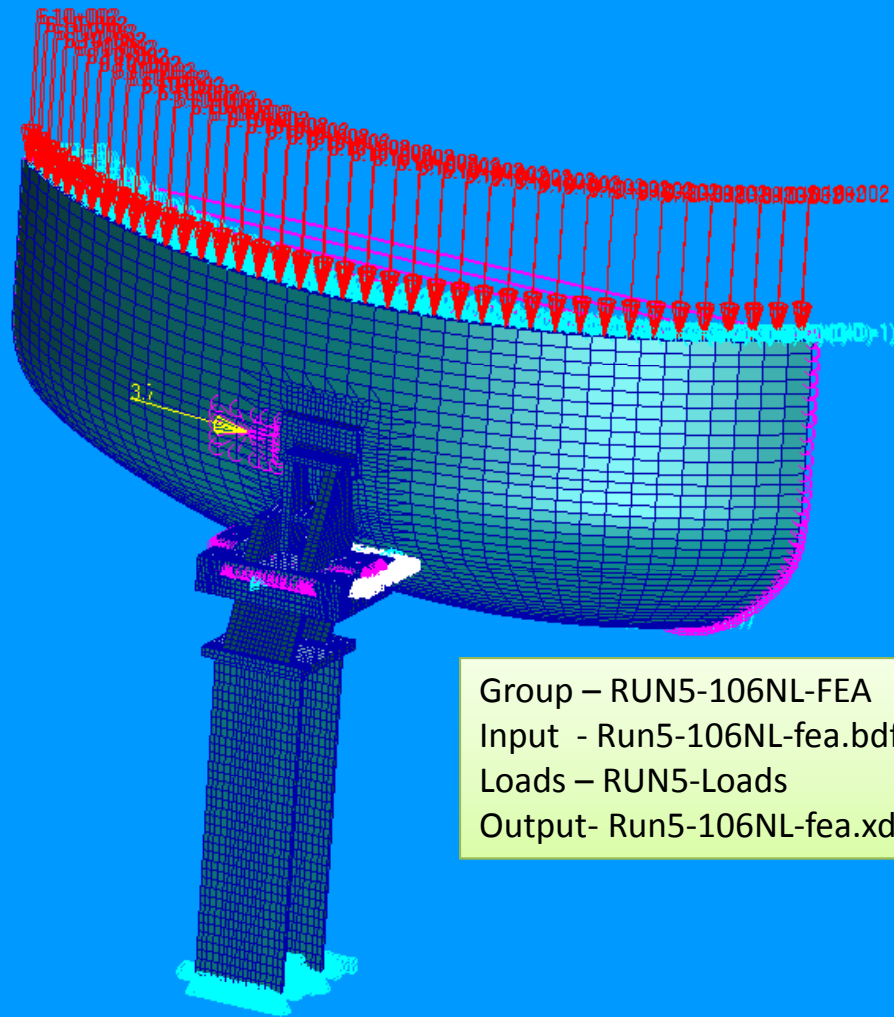
Changes for RUN #5



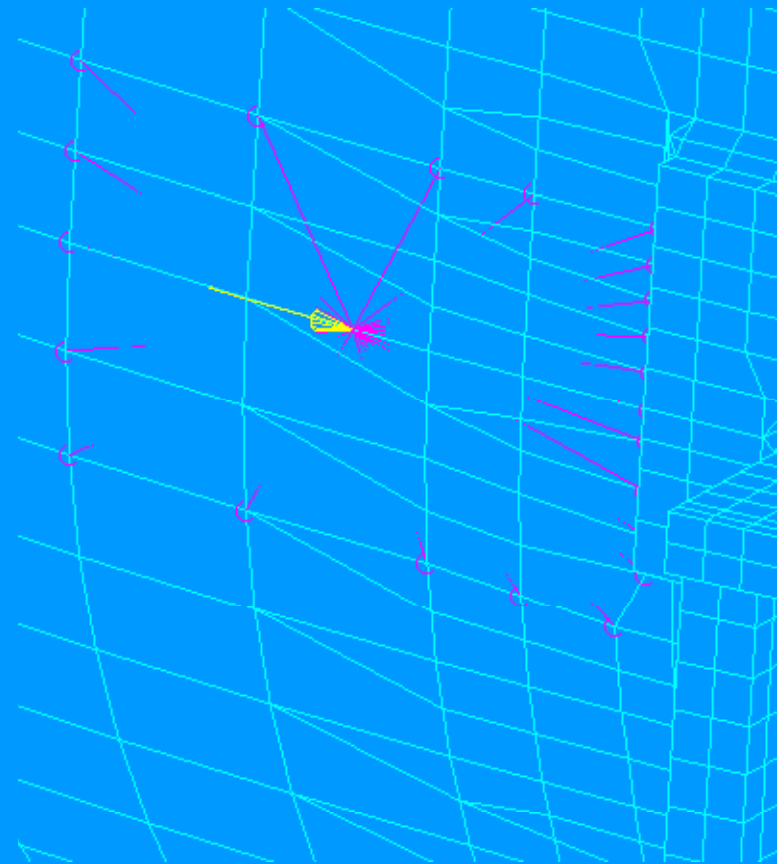
Introduction of the near-by Clevis force and its application is added to the Run #4 simulation. The VV-chair was redesigned because of the interference created by the new TF coil link connections to the VV. Therefore, the clevis is placed very close to the VV-chair structure. The slides on the left show this closeness and the RIGID BODY formation to simulate this condition. This body consists of the central node to which outside nodes are connected rigidly. Therefore, load applied at the central node distributes the force to the dependent nodes on the outside nodes which describe the maximum dimensions of the welded clevis. A force equal to 37,000. lbs is applied at the central node using cylindrical Coordinate System #1 in the Theta – direction.

Central node is independent.

RUN #5 - Configuration



Group – RUN5-106NL-FEA
Input - Run5-106NL-fea.bdf
Loads – RUN5-Loads
Output- Run5-106NL-fea.xdb



RUN #5 is a Sol 106 (Non-Linear) run with B.C. and Load from Run #4 and additional rigid body simulation for the Clevis pad which resists the 37,000. lbs shear load parallel to the VV surface. (Yellow arrow)

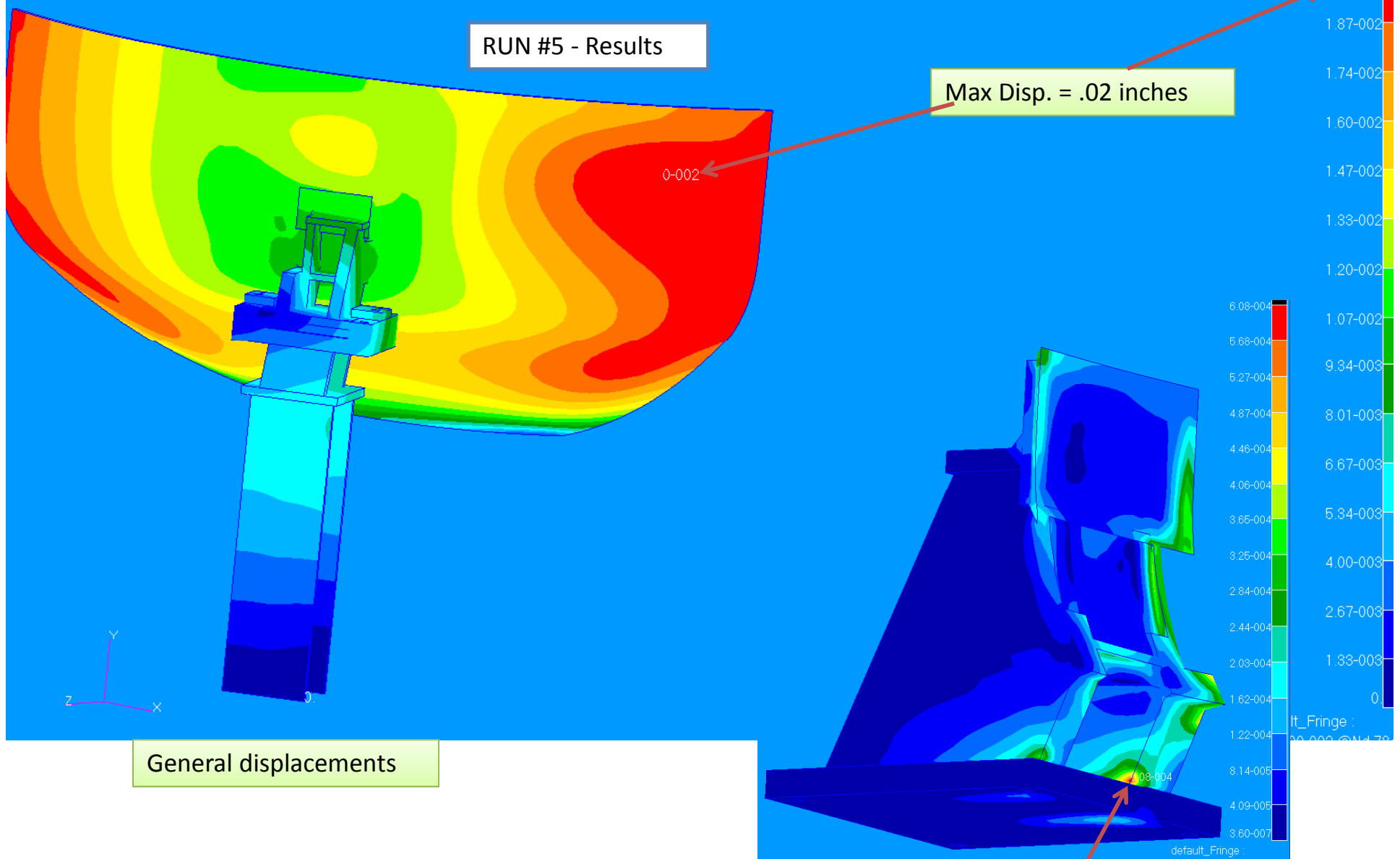
RUN #5 - Results

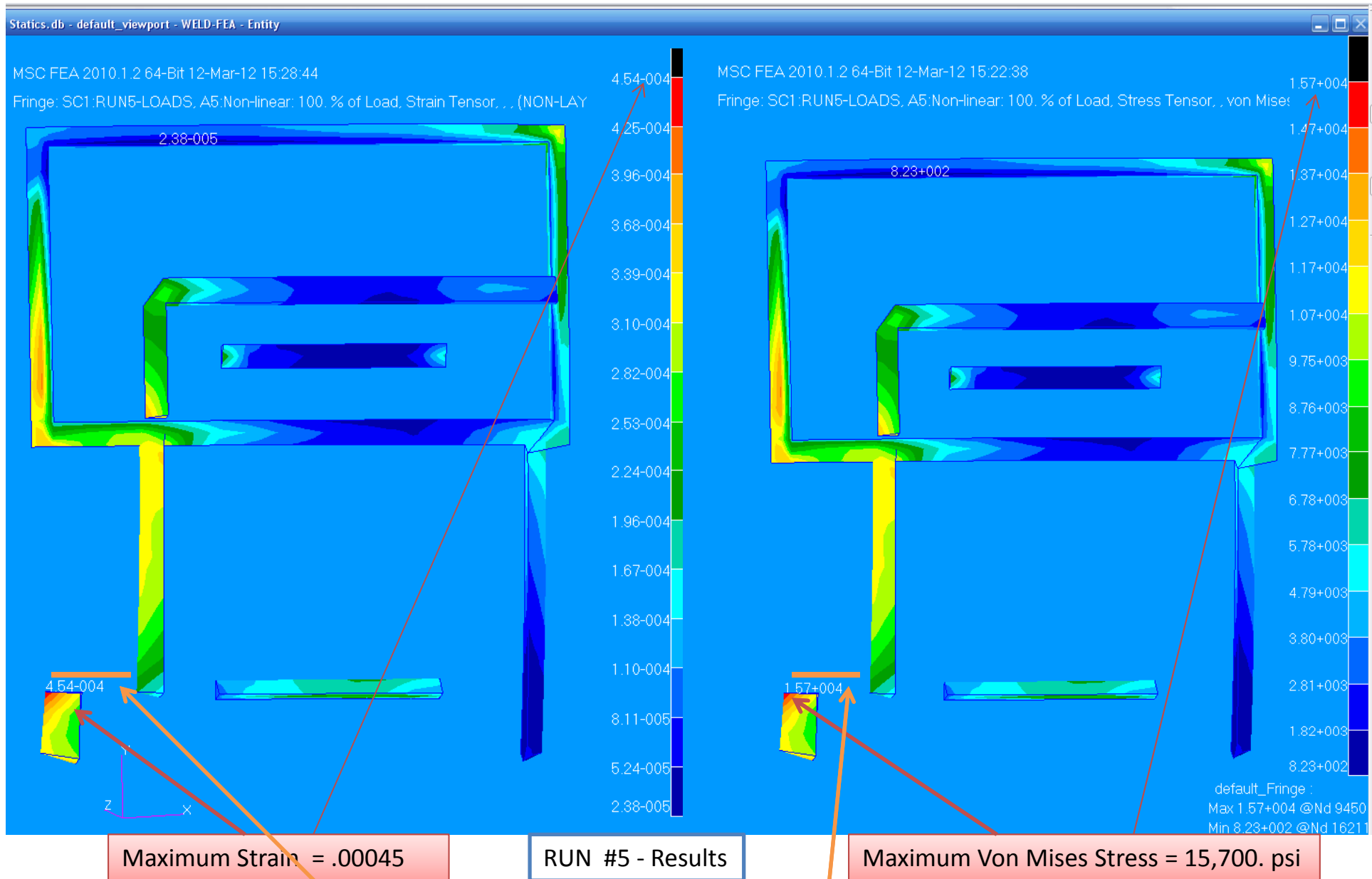
Max Disp. = .02 inches

0-002

General displacements

Maximum strain = .000608





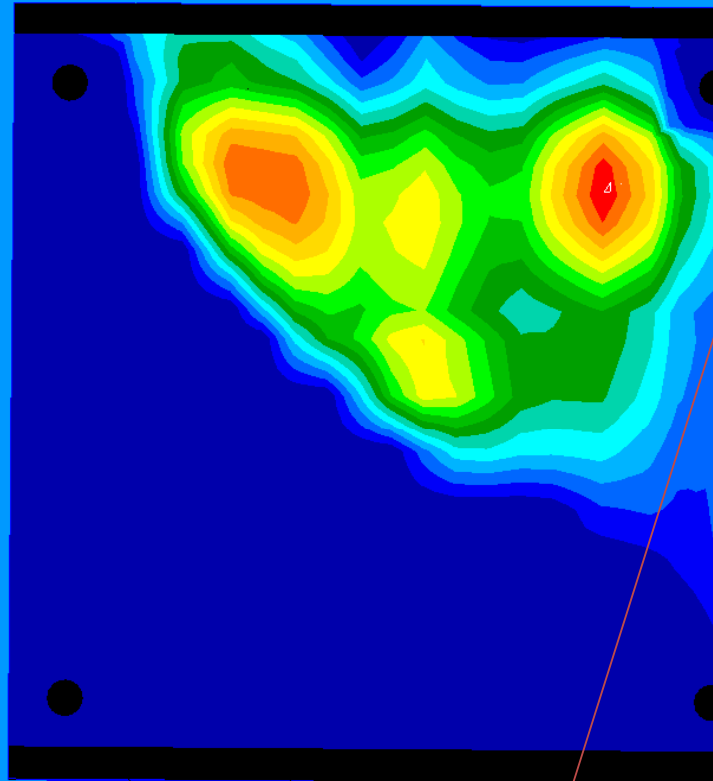
MSC FEA 2010.1.2 64-Bit 12-Mar-12 15:35:42

10.1.2 64-Bit 12-Mar-12 15:39:43

Fringe: SC1:RUN5-LOADS, A5:Non-linear: 100. % of Load, Gap Results, Force, Magnitude,

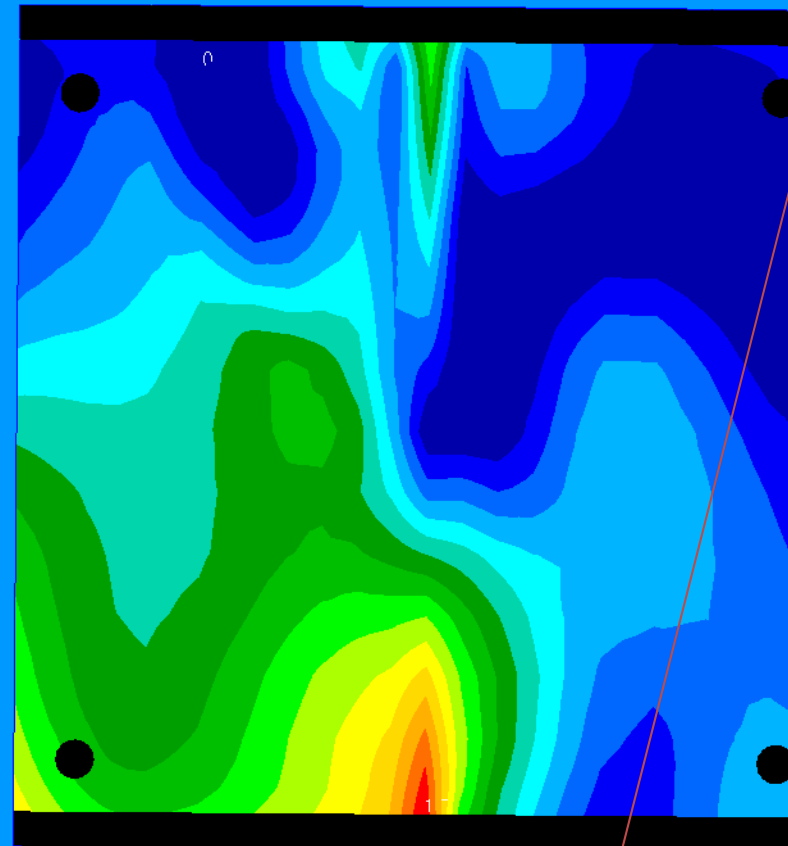
Fringe: SC1:RUN5-LOADS, A5:Non-linear: 100. % of Load, Gap Results, Slip, Magnitude, At t

RUN #5 - Results



Maximum = 414. lbs

Force distribution through the Gaps.
Note the difference with the Run #4 results (Slide 21). Forces tend to shift to the right, since, the 37,000. lbs force at the Clevis is in that direction.

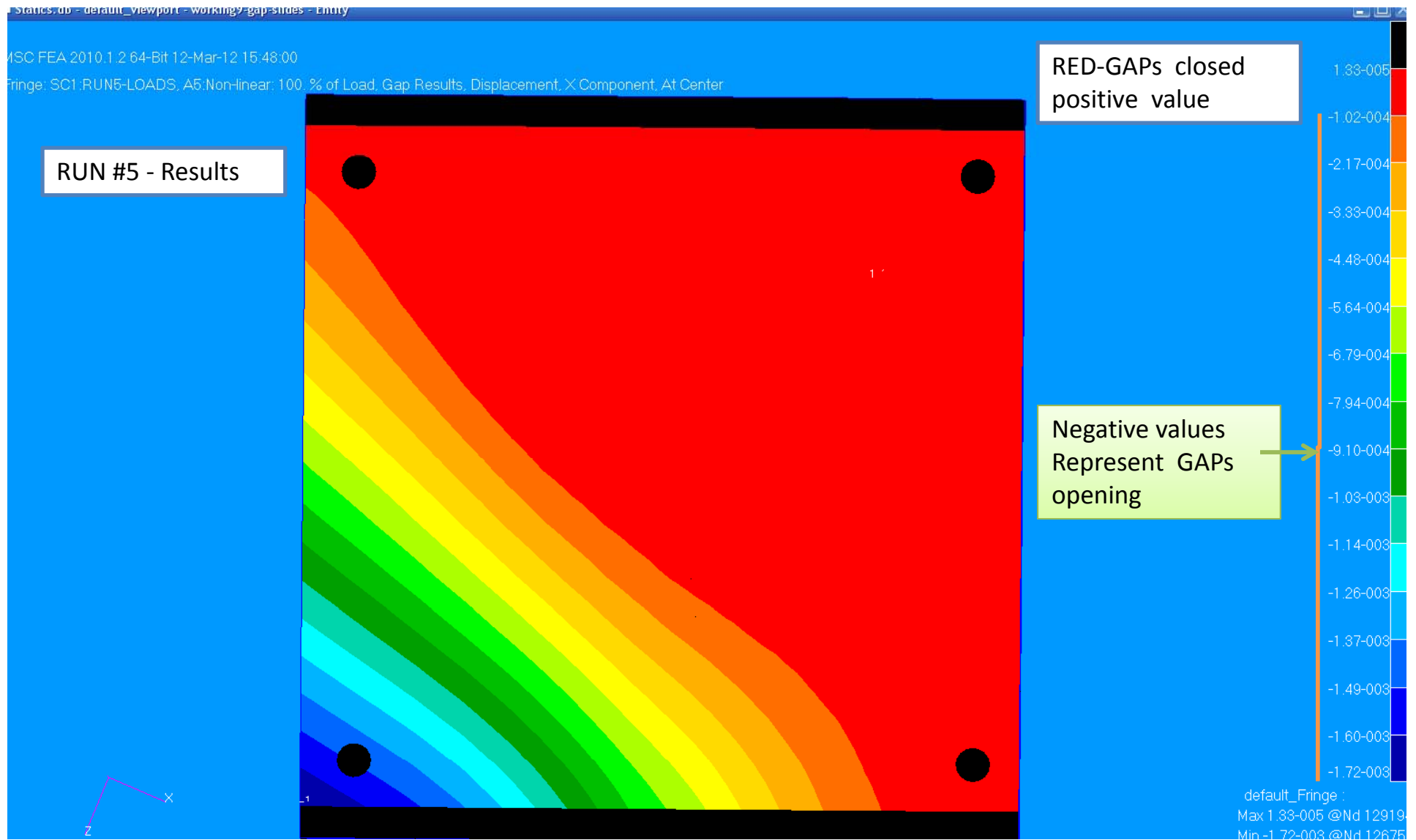


Maximum slide = .00018 inches

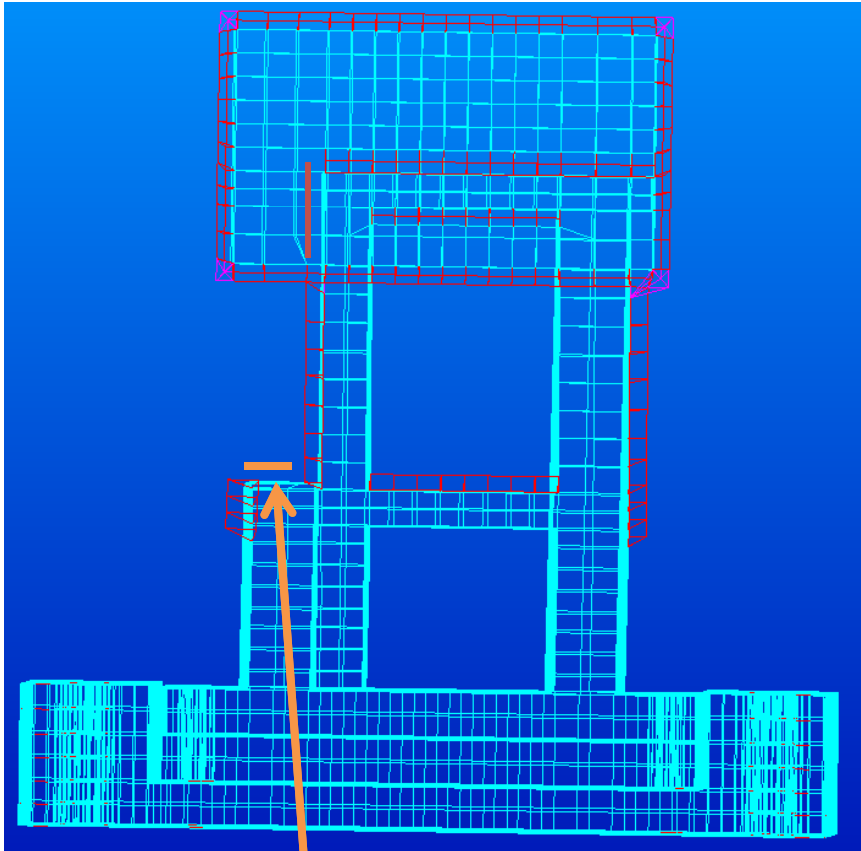
GAPs sliding profile. Compare with RUN #4 results (Slide 21).

Fringe :
+002 @Nd 12919
005 @Nd 131584

28
ge :
4 @Nd 130019



Coordinate 2 controls (defines) the used GAPS. The X-direction (normal to page) is the principal gap direction which defines if the gap is open or closed. Please see the ".bdf" input file for details.
(Compare with Run #4 results shown on Slide 22.)



Area that would be extremely desirable to weld, but, may not be readily accessible

Red elements represent the actual welds

Welding scenario for the redesigned VV-Chair support

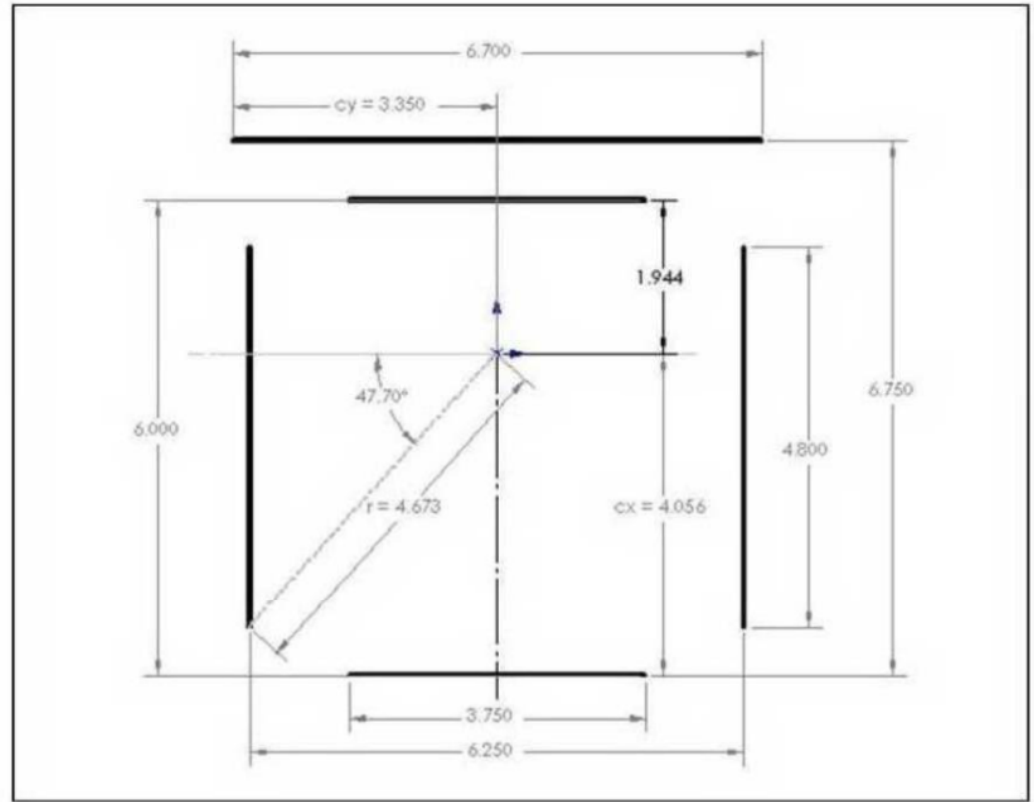
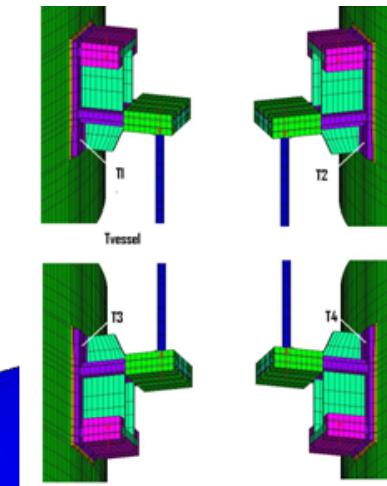
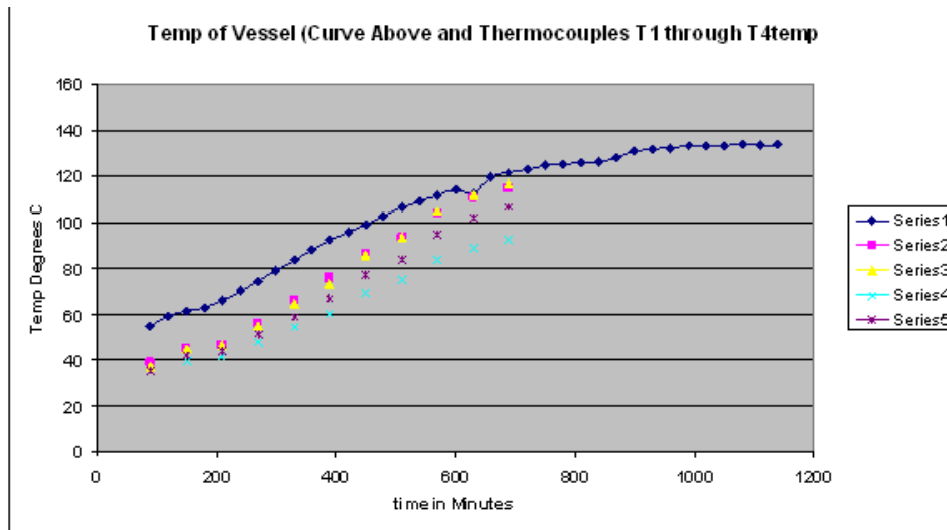


Figure 2 - Support Leg-to-Vessel Weld Dimensions

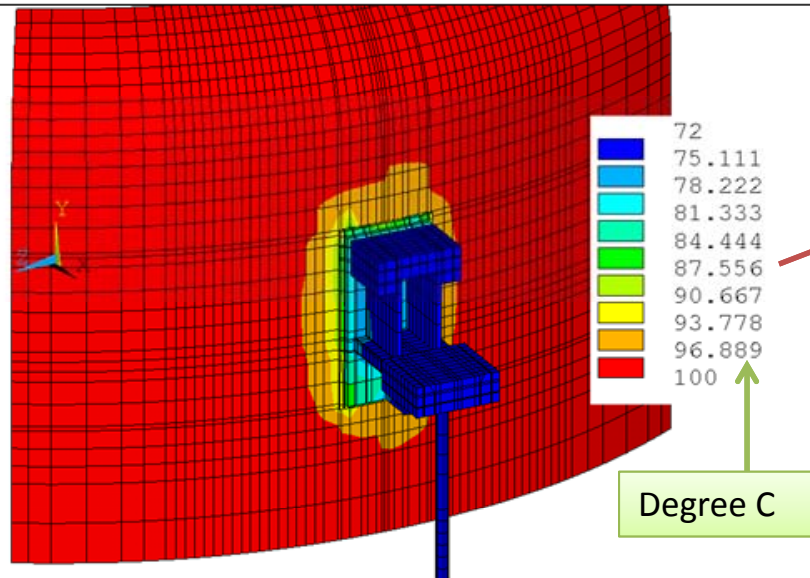
Original weld configuration – standard chair
Note: received from M. Smith (E-mail)



RUN #6 Bakeout

Conversion to
degrees F

C	F
75.1	167.20
78.2	172.76
81.3	178.34
84.4	183.92
87.6	189.68
90.7	195.26
93.8	200.84
96.9	206.42
100.	212.0



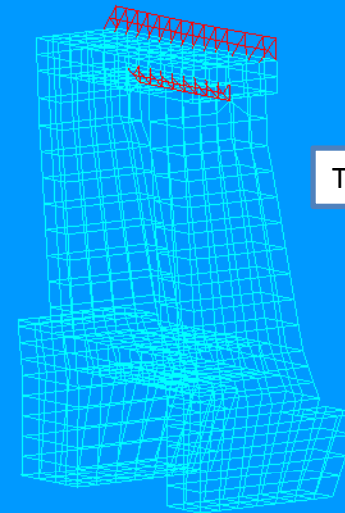
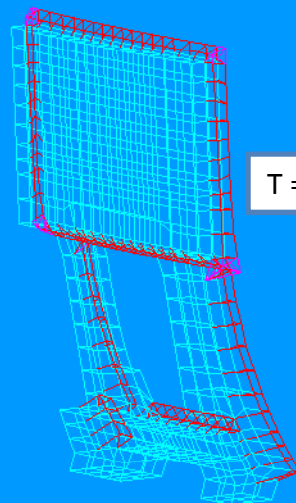
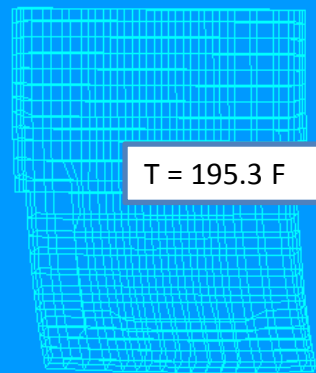
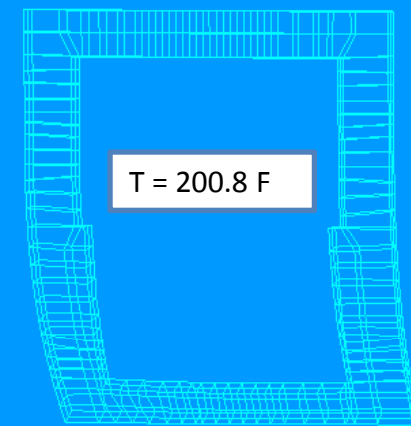
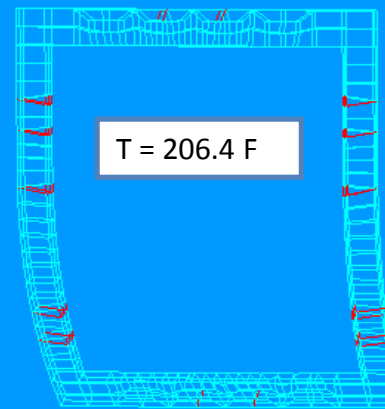
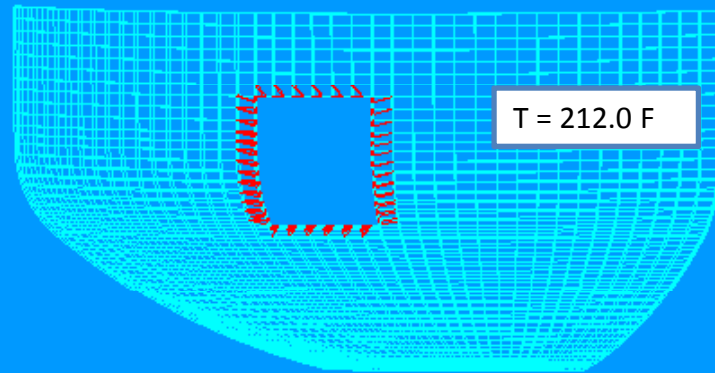
Degree C

SEQV (AVG)
PowerGraphics
EFACET=1
AVRES=Mat
DMX =.008254
SMN =207221
SMX =.162E+09
207221
.181E+08
.361E+08
.540E+08
.719E+08
.898E+08
.108E+09
.126E+09
.144E+09
.162E+09

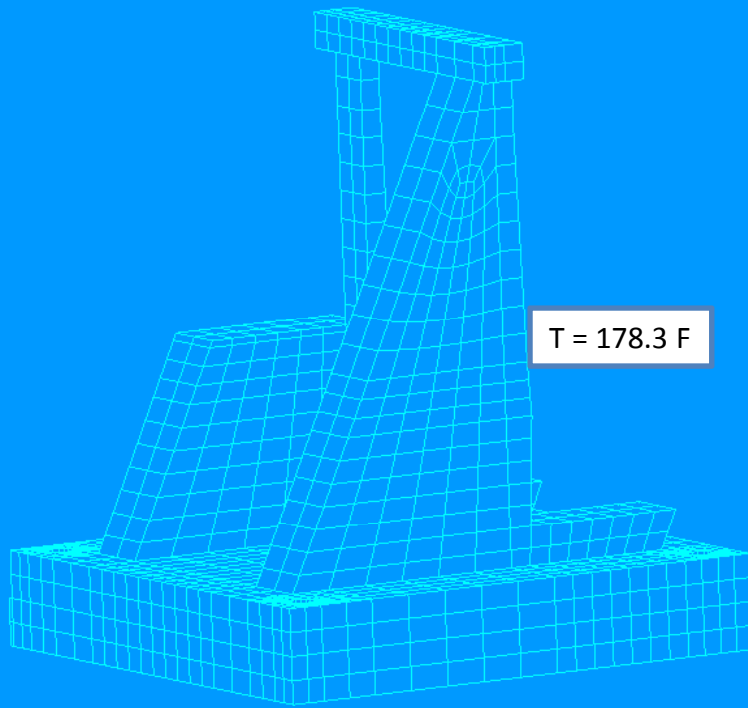
Use this for RUN #6

This data is taken from the previous PF 4 and 5 support hardware simulation. Similar temperature scenario exists in the area of contact between the VV and the main support Chair structure.
This temperature data has to be converted to degrees F in order to be used in the present RUN #6 simulation.

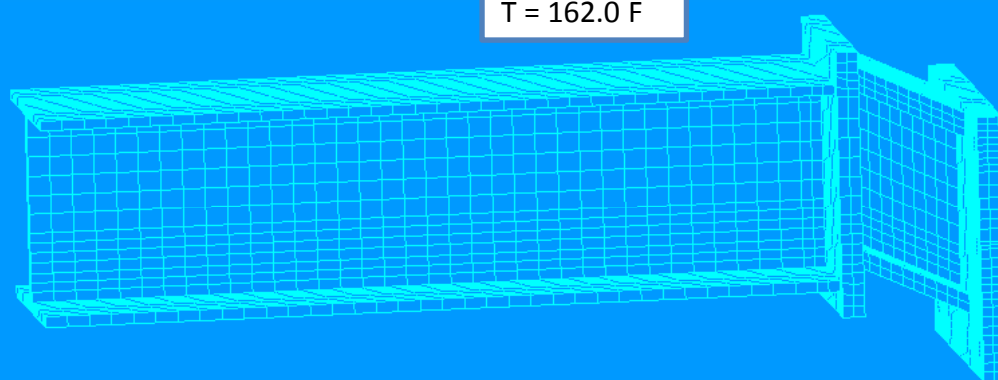
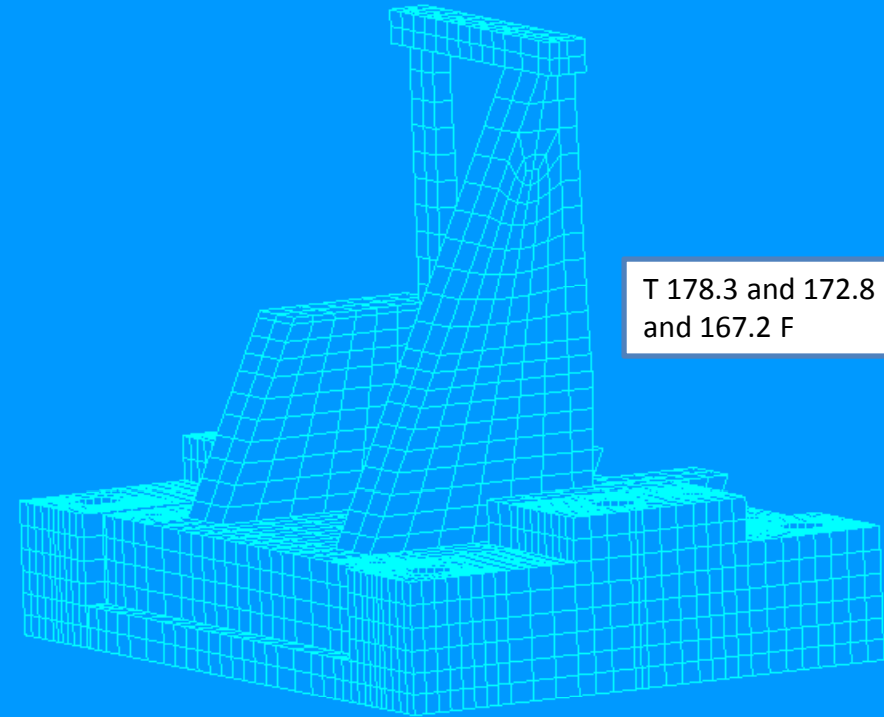
RUN #6



Applied temperature distribution according to the converted table from the previous slide. These groups are most critical in order to determine the VV bake out effects. This is a highly approximated temperature profile. The VV support structure is a very effective heat sink. Because of its total mass, the gradients are probably reasonable. If better simulation is necessary, then, a complete thermal analysis would be required to determine temperature at each nodal point (time consuming procedure).



RUN #6



Please Note: The described temperature distribution is the result of the author's assumed interpolation.

RUN #6 – With Temperature

```
$ Nonlinear Static Analysis, Database
SOL 106
CEND
TITLE = MSC.NASTRAN JOB CREATED ON 17-FEB-12 AT 08:48:38
ECHO = NONE
MPC = 20134
TEMPERATURE(INITIAL) = 10
SUBCASE 1
    TITLE=106 with THERMO
    SUBTITLE=RUN6-Loads
    NLPARM = 1
    SPC = 2
    LOAD = 12
    TEMPERATURE(LOAD) = 2
    DISPLACEMENT(SORT1,REAL)=ALL
    SPCFORCES(SORT1,REAL)=ALL
    STRAIN(SORT1,REAL,VONMISES,STRCUR,BILIN)=ALL
    STRESS(SORT1,REAL,VONMISES,BILIN)=ALL
$ Direct Text Input for this Subcase
BEGIN BULK
$ Direct Text Input for Bulk Data
PARAM      POST      0
PARAM      AUTOSPC YES
PARAM      PRTMAXIM YES
NLPARM      1      10      AUTO      5      25      NO
$ Elements and Element Properties for region : HexwedTet
```

From the "RUN6.bdf" file

Group – RUN6-106HL-FEA-thermo
 Input -- RUN6-NL-THLoads.bdf
 Loads -- RUN6-Loads
 Output – RUN6-NL-THLoads.xdb

RUN #6 Loads formulation is as follows:

Mechanical loads and B.C. are identical to those applied in RUN #4, plus the described temperature distribution scenarios on the previous slides.

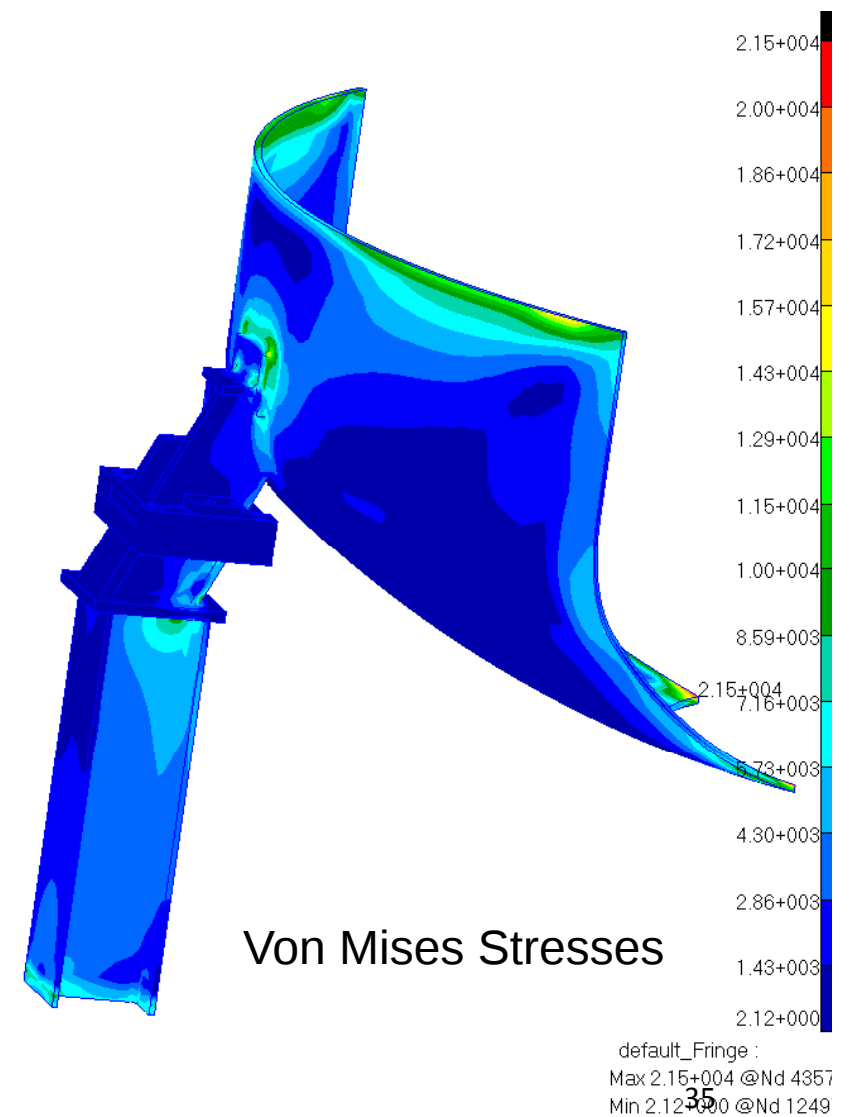
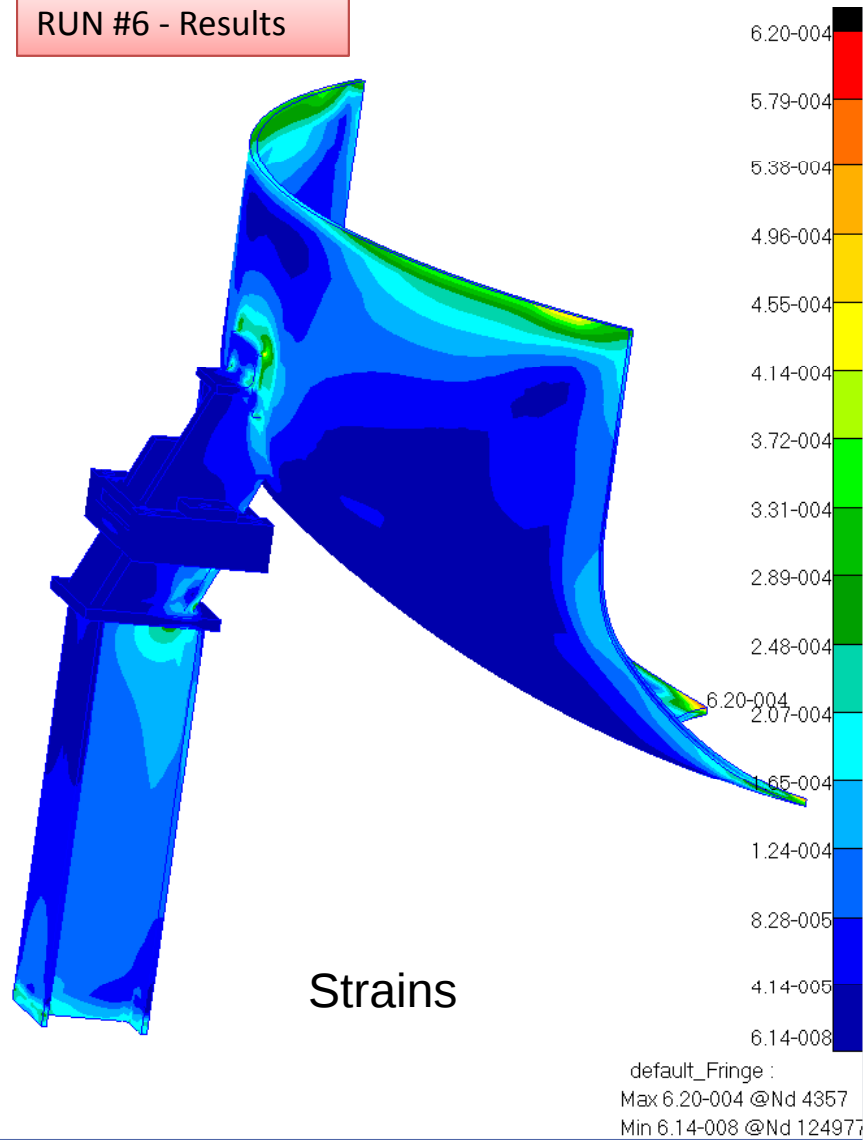
MSC FEA 2010.1.2 64-Bit 16-Mar-12 14:47:52

Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Strain Tensor, . .

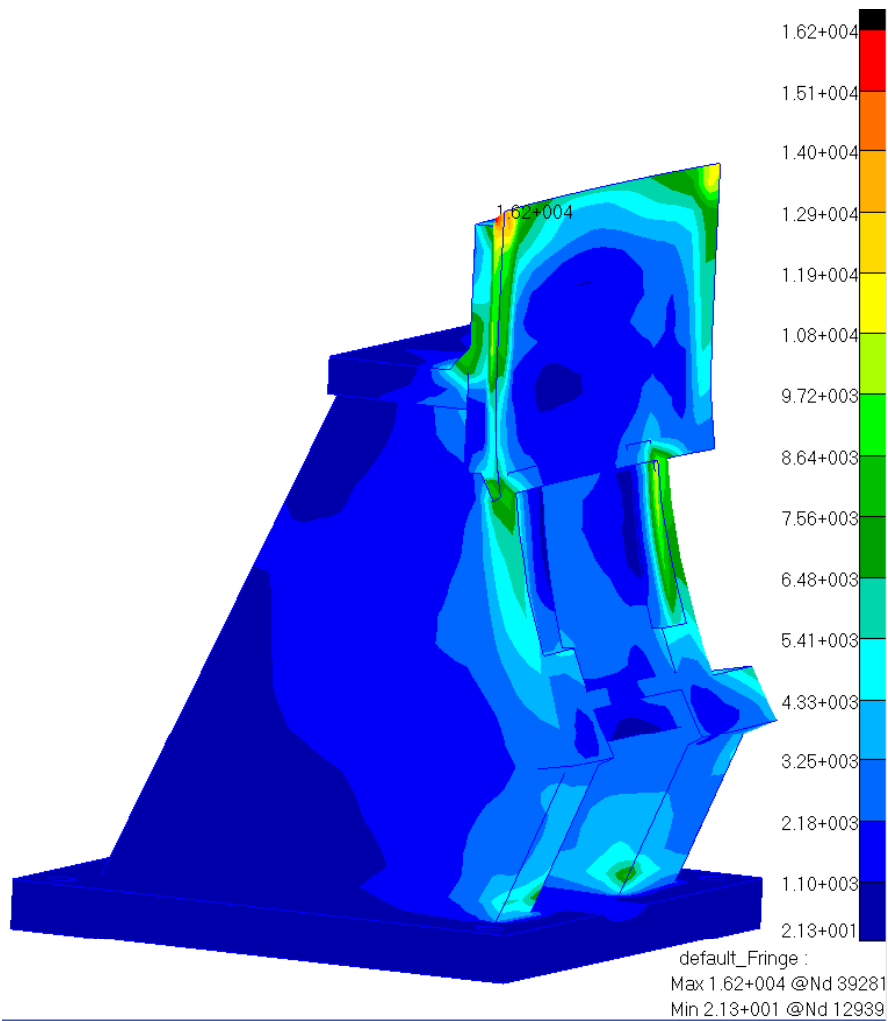
MSC FEA 2010.1.2 64-Bit 16-Mar-12 14:44:43

Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Stress Tensor, . von Mises, (NON-LAYERED)

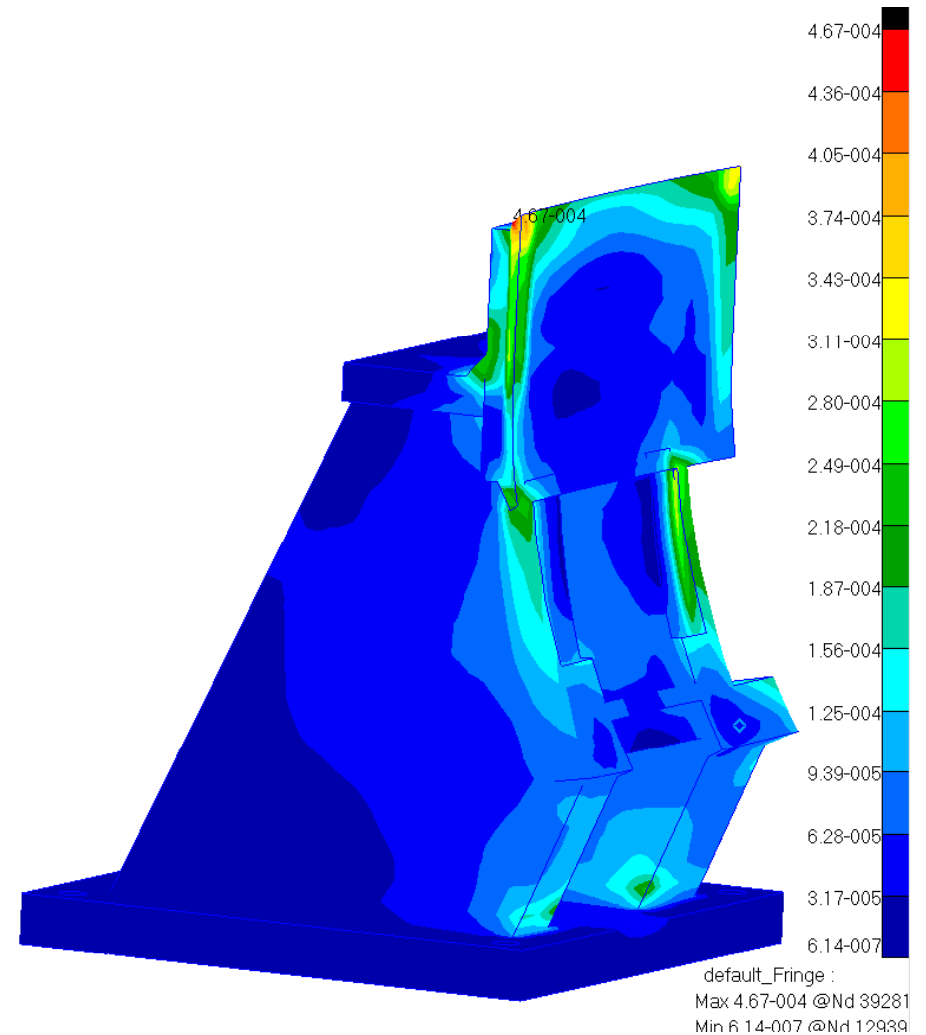
RUN #6 - Results



RUN #6 - Results



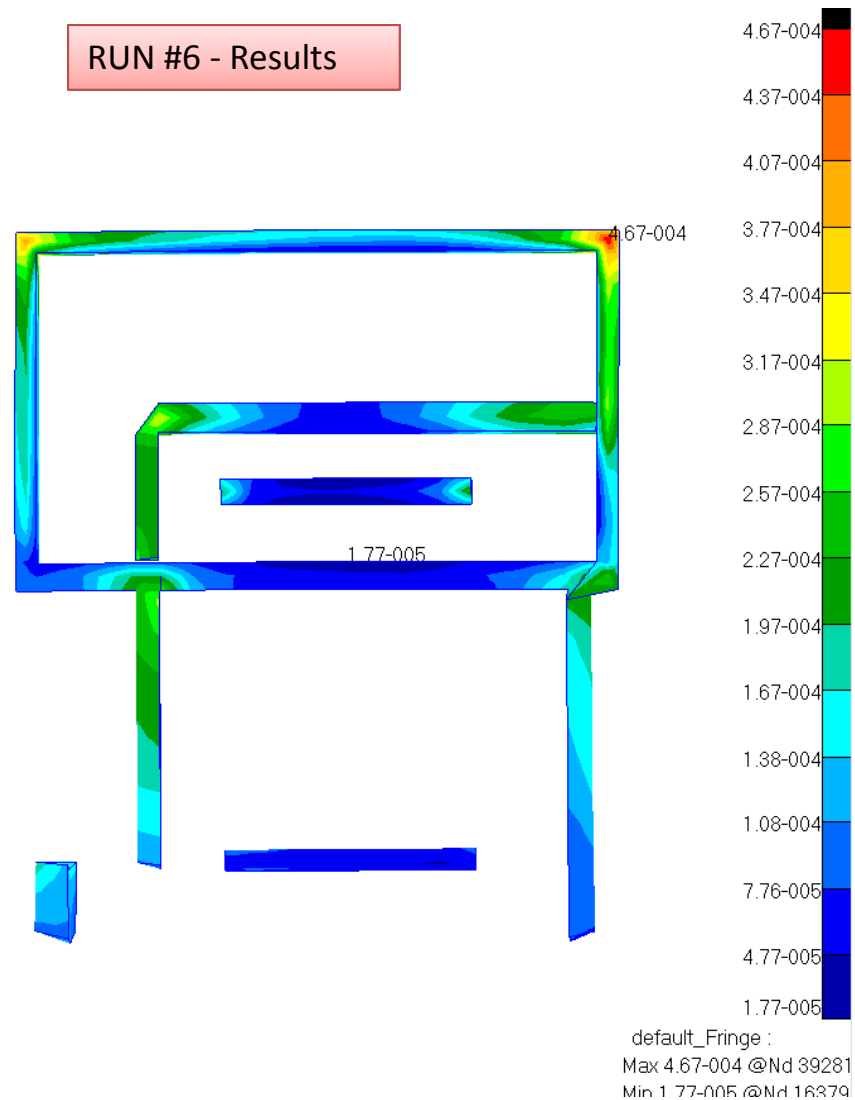
Von Mises Stresses



Strains

MSC FEA 2010.1.2 64-Bit 16-Mar-12 15:22:36

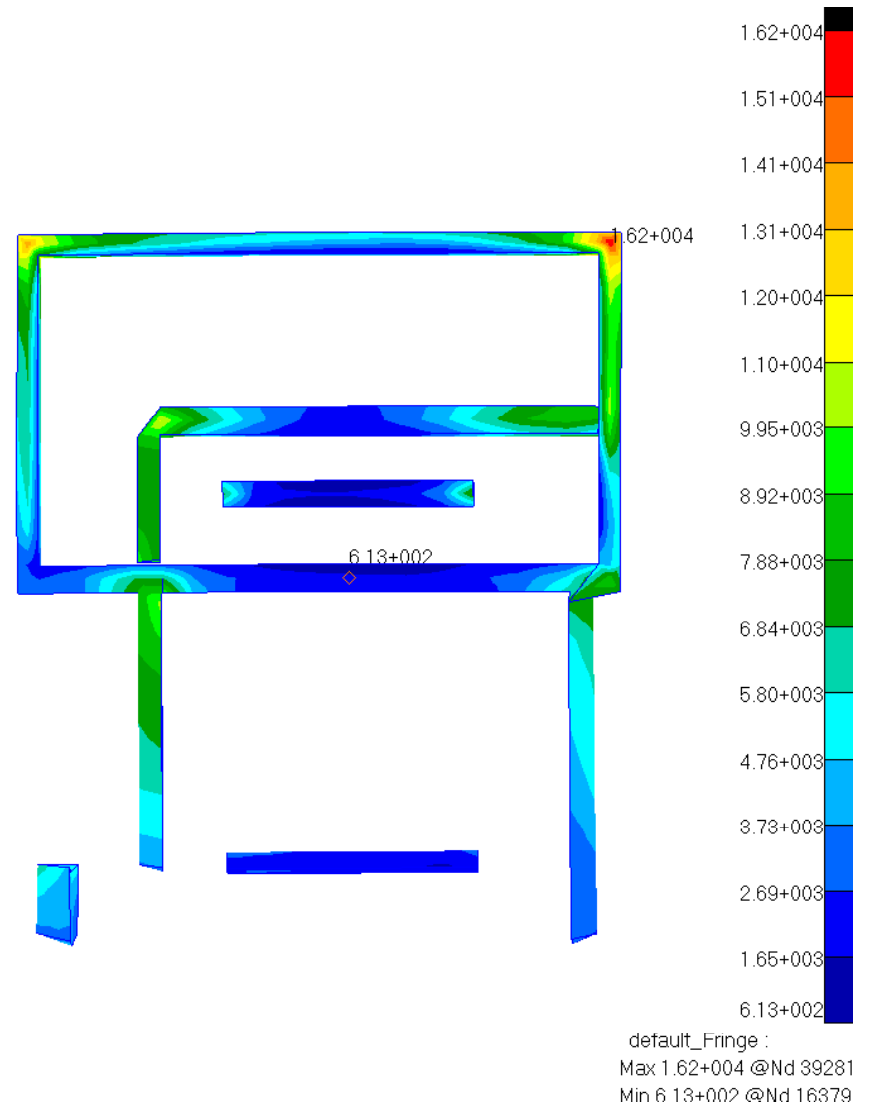
Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Strain Tensor, . .



Strains

MSC FEA 2010.1.2 64-Bit 16-Mar-12 15:17:01

Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Stress Tensor, , von Mises



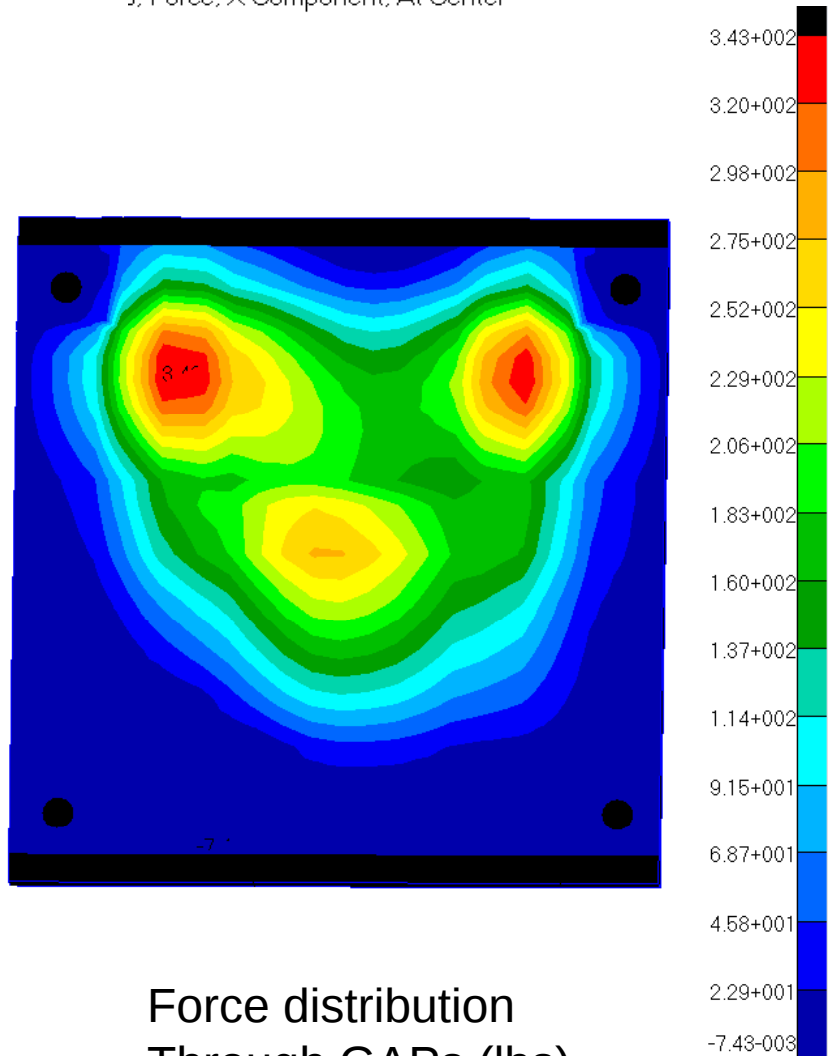
Von Mises Stresses

MSC FEA 2010.1.2 64-Bit 16-Mar-12 15:37:23

RUN #6 - Results

Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Gap Results, f

s, Force, X Component, At Center



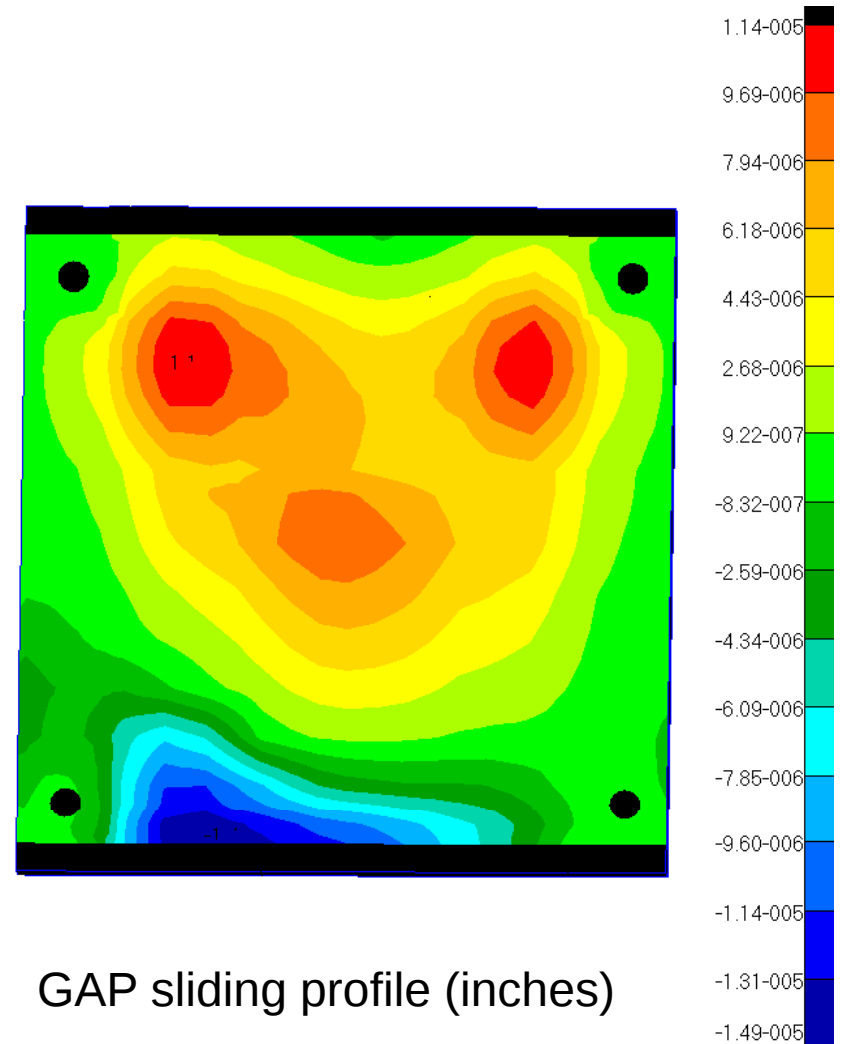
Force distribution
Through GAPS (lbs)

default_Fringe :
Max 3.43+002 @Nd 128121
Min -7.43-003 @Nd 128752

MSC FEA 2010.1.2 64-Bit 16-Mar-12 15:28:41

Fringe: RUN6-Loads, A7:Non-linear: 100. % of Load, Gap Results,

s, Displacement, X Component, At Center



GAP sliding profile (inches)

default_Fringe :
Max 1.14-005 @Nd 128121
Min -1.49-005 @Nd 128752

Complete Summary explanation for all Runs

From the examinations of the six different FEA simulations presented by the above slides, the following evaluation can be stated:

- Basic FEA geometry simulations (nodal points, elements), for all runs, are the identical.
- The differences are in the applications of the model support/boundary conditions (SPCs) and their locations for each specific run. This was implemented in order to fully investigate the worst possible magnitude and locations of stresses and strains in the critical areas (especially the welded locations between the VV and the support structure).
- Runs #1, #2 and #3 are fully linear (the sliding effects were not considered). GAPS were replaced with RBE2 rigid elements. Runs #4, #5, #6 are non linear (NASTRAN Solution 106) where GAPS were fully utilized to evaluate the contributions due to the VV-Chair sliding effects. For these runs, GAP element contours are included to evaluate the correctness of the load transmission at the sliding surface (see the appropriate slides).
- Runs #1 and #2 were included here only for model verification and to demonstrate the effects of some extreme boundary support conditions which are used to improve the total model simulations.

In general, the following can be concluded and recommended for minimizing stresses and strains: For Run #1 under the most severe boundary conditions, the maximum stresses in the weld locations are equal to 15,300. psi. For Run #2 (less severe boundary condition) maximum weld stress decrease to 10,200. psi. From these two runs, the maximum possible stresses are established for total VV dead weight effects. Inclusion of the additional static loads may change the locations as well as the stress/strain values.

Complete Summary explanation for all Runs (cont.)

Welds Maximum

Chair Body Maximum

Run #	VM Stress	Strain	VM Stress	Strain
#1	15,300. psi	.0004	56,700.*	.0002
#2	10,200.	.0005	12,500.	.0005
#3	10,500.	.0003	15,500.	.000444
#4	11,100.	.00032	15,200.	.000438
#5	15,700.	.00045	21,100.	.00061
#6	16,200.	.00047	21,500.	.00062

* This stress is high because of the conservative boundary conditions used and should be disregarded.
(see RUN #1 results slides)

Final conclusion - Recommendations

Stainless steel with a yield stress of 42,000 psi was used for these analyses. Comparing to the above table, all the stress and strains are well within allowable limits (with the exception of Run #1 as noted above) and provides an adequate static margin for welds and the support structure alike.

Strong recommendation is to weld corner locations carefully in order to minimize the stress concentrations there (see Slide 30). It is advised that regular inspections of these locations should be performed.

Appendix A

All the data is saved in the following locations and it includes: the write up, analyses data base which includes all the analyses groups, input files for all completed runs (.bdf files), and the output results (.xdb files).

Folder name is “Vessel-Support-Chair-Data”

P:\public\Snap-srv\progoff

G:\Nastran P.R. - (external drive on ASALEHZ-64PC)