NSTX Upgrade

TF Lead Flag Extensions & Connectors

NSTXU-CALC-132-15-00

Rev. 0

December 2014



Prepared By:

P. Titus

A. Brooks

Reviewed By:

I. Zatz,

| Calculation # | <u>NSTX-CALC-132-15-00</u> | Revision # | <u>0</u> | WP #, if any (ENG-032) | <u>1672</u> |
|---|---|--|---|--|--|
| Purpose of Calc | culation: (Define why the calculation | is being performed. | .) | | |
| The purpos as they hav documente iterated suc extracted for been altered | se of this calculation is to quali ve been manufactured and inst d in calculation NSTX-CALC ch that the area of interest has or the area of interest, specifica d and the joints joining the plate | fy the NSTX-U alled. These we -132-06-00. S changed. The c illy, the shapes es are to be EB | TF lead flag ere originally ince that tim original analy of the plates welded rathe | extensions and oute analyzed as part of e, the design of the sis was resurrected a that form the connec r than formed from a | r TF connector a larger mod parts has been and results we tors which has solid. |
| References (List | any source of design information inclu (See the | ding computer prog e Body of the C | ram titles and realculation) | vision levels.) | |
| Assumptions (Id | dentify all assumptions made as part of | this calculation.) | | | |
| Per Referen | nce [2] | | | | |
| Calculation (Cal | culation is either documented here or a | ttached) | | | |
| Analysis re | esults included as part of this do | ocument. | | | |
| Conclusion (Spe | ecify whether or not the purpose of the c | calculation was acco | omplished.) | | |
| The stresse design base are found t effective or minimum s full power replaced w conservativ lead to an o of the flex connectors replacement fracture me usage facto | es in the electron beam welded j ed on the analysis by Tom Will to be within allowables. The 'C rack at the back side of the well section that does not satisfy the shots. This will be acceptable with a part with a full section we estimate of the R value. A B extended life for the type "C" and connector assembly, plus is undertaken, a new global at of connector type "C" is re- cechanics assessment. Miners Ru- pr well below 1.0. | joints in connec lard and sub mo C' connector joi ld. This was ma e full life requir for first year on and full secti prooks has point connector. Anal s sub models o model, with the ecommended. The ule calculations | tors 'A', 'B' odels by A. B int had a part ichined away rement for the perations, but ion welds. T ted out that a lysis is based f the connec ne proper geo The existing based on the | and 'C' are estimate rooks. The "A" and ial weld specified w to leave a smooth su e part. The life is es t the "C" connector his recommendation better estimate of th on use of T. Willard tors. If another anal ometry updates is re- connector might be e first year TF shot s | d for the curre "B" connecto hich left a larg irface, but left timated as 200 will need to b is based on e R value cou l's global mod ysis of the fla equired. Whi qualified by pectrum show |
| Cognizant Engi | neer's printed name, signature, | and date | | | |
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Checker's printed name, signature, and date

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| Rev 0 | Initial Issue |
|-------|---------------|

4.0 Executive Summary

The outer TF connector design was updated in October 2013. A radial position error or discrepancy between as-builts and the CAD model caused a redesign of the connectors to gain back a 1 inch interference. The original design employed a solid plate and it was difficult to find a solid piece of CuCrZr from which to cut the dog-legged geometry. An e-beam weld jointed plate design was chosen. This was determined to be the optimal way to preserve the strength and integrity of the CuCrZr material in the joint and would be superior to brazing or conventionally welding the joint. The design changes are documented in drawings E-DC1456 thru E-DC1460 Rev. 2 (Appendix B), which apply to connector type 'A' thru 'E', respectively. The consistent ECN is #7134.

The stresses in the electron beam welded joints in connectors 'A', 'B' and 'C' are estimated for the current design based on Tom Willard's analysis and sub models by A Brooks. Two of the three joints are found to be within allowables. The 'C' connector joint,

with a nominal full thickness was the most highly stressed at approximately 20 ksi. However the "C" joint was fabricated with a partial penetration e-beam weld that left a large effective crack on the backside of the weld.

Tom Willard originally modeled the entire TF joint assembly in NSTX-CALC-132-06-01 [2], which was completed in 2011. At that time, the focus of the analysis was on the TF strap that connected to the inner leg. Later calculations included the hardware connection to the outer leg. The TF strap assembly fingers were reviewed in NSTX-CALC-132-14-00 [3]. That report was based on the Tom Willard peer review from February 2013, and included the analysis results of the TF outer leg flag-to-lead bolted joint design. Also in February 2013, there was a peer review of the design of TF lead extensions and support brackets (Appendix A). Willard updated his model to include the proposed design of these components as they existed at that time. However, analytical results for the lead extensions and support brackets were never documented. This report reviews the results recently extracted from the Willard models pertaining to the lead flag extensions and connectors. In addition, it was observed that the design has been iterated since the model was last updated.

The outer TF connector design was updated in October 2013 to reflect the decision to electron beam weld the joints. This was determined to be the optimal way to preserve the strength and integrity of the CuCrZr material in the joint and would be superior to brazing or conventionally welding the joint.

Other components of the connection to the outer legs of the TF are in addressed in other calculations. The fingers that support the extensions are included in "NSTX Upgrade TF Strap Assembly Fingers" NSTXU-CALC-132-14-00 ref [3].

5.0 Input to Digital Coil Protection System

The lead flag extensions and connectors share the same loads as the flex connector. A DCPS algorithm for the flex, based on poloidal and toroidal field magnitudes is planned, and this will also limit the loads on the leads and connectors to the levels computed in this calculation. The Type "C" connector is fatigue limited and a cycle counting/Minors Rule procedure that is described in section 8.3.3 will be needed, separate from the DCPS.

6.0 Design Input 6.1 Criteria

The Criteria for this calculation are contained in the NSTX-U Structural Design Criteria, Ref [6]

6.2 References

[1] Drawings E-DC1456 thru E-DC1460 (all Rev. 2) [Appendix B]
[2] NSTX-CALC-132-06-01 TF Flex Joint & TF Bundle Stub, T. Willard.
[3] NSTX-CALC-132-14-00 TF Strap Assembly Fingers, L. Dudek. December 11, 2013
[4] email from Larry Dudek to Erik Perry Oct 10 2014
Erik,
All of the lead extensions will require some cleanup. The worst ones are the E-DC1458's. I have a separate sketch attached showing what is needed there. they will need to be cut in two places as shown on the sketch. The rest (Parts E-DC1456,57,59 & 1460) just need to have the 1/8" tab ground (or milled) off and make the machined surface smooth. I have attached the sketches and the original drawings for reference. Lew has the sketches to incorporate them into a new revision to formally

document the work.

Let me know when you are ready to begin, I would like to see the first of each type to inspect. Thanks,

Larry

[5] email from Stefan Gerhardt, Oct 2 2014:

Stefan Gerhardt <sgerhard@pppl.gov> Oct 2 to James, Steve, Larry, Arthur, me, Masayuki, Jonathan

Guys,

These below in blue are some assumptions about the TF usage in the first year. My bosses are happy (enough) with this.

I would think that if 2000 pulses at 1.0 T are qualified, then 2000 pulses at less than 1.0 T are OK?

But I do wonder what assumptions were made on the background magnetic field that gives the JxB force. For instance, a VDE and associated current response could lead to higher background fields? So I wonder if we need another DCPS algorithm?

Let me know if you need more/different.

•16 run weeks, 5 days/week, 8 hours/day, 3 shots/hour = ~2000 shots (1920 in reality)

•We stated in the FWPs that we would go at high as 0.8 T, at least on occasion

•We will commission operations at 0.55 tesla.

•Operation beneath 0.45 T will be very limited.

•CHI and RF will want the highest TF that they are allowed to use.

•Shot spectrum to assume:

 \circ 5% of shots at 0.45 T = 100 shots

 $\circ 30\%$ of shots at 0.55 T = 600 shots

 $\circ 25\%$ of shots at 0.60 T = 500 shots

 $\circ 25\%$ of shots at 0.70 T = 500 shots

 $\circ 15\%$ of shots at 0.80 T = 300 shots

[6] NSTX Structural Design Criteria Document, NSTX_DesCrit_IZ_080103.doc, Feb 2010 I. Zatz

[7] NSTX-U Design Point Spreadsheet <u>http://w3.pppl.gov/~neumeyer/NSTX_CSU/Design_Point.html</u>, C. Neumeyer

6.3 Photos and Drawing Excerpts

Design drawings are included in Appendix B. The sketches below provide the details for the weld clean-up.



Figure 6.3.-1 Weld Clean-Up for types "A" and "B", Ref [4]



Figure 6.3.-2 Weld Clean-Up for types "C", Ref [4]



Figure 6.3.-3 Type "C" Before Weld Clean-Up





Figure 6.3.-4 Type "C" After Weld Clean-Up

6.4 Materials and Allowables

The outer TF connector was specified to be CuCrZr by T. Willard in ref 1. The design was updated in October 2013 to reflect the decision to electron beam weld the joints. This was determined to be the optimal way to preserve the strength and integrity of the CuCrZr material in the joint and would be superior to brazing or conventionally welding the joint.



Figure 6.4-1 CuCrZr SN Data

The mill Certs for the CuCrZr used for the extensions are included in Appendix "D". A 52 ksi yield and 65.5 ksi ultimate are reported.



Figure 6.4-2 CuCrZr Charpy Impact Data



Fig. 2. Ultimate tensile strength (S_u) and yield strength (S_y) of CuCrZr alloy in SAA condition and minimum tensile strengths. Data points are from the ITER MPH database.

Figure 6.4-3 CuCrZr Physicals Sult and Svield

7.0 Models

Tom Willard's model includes connector types 'A', 'B' and 'C' as they were designed in 2011. In examining the most up-to-date Willard model (Figures 7-1 & 7-2), it is evident that the focus of the analysis, as documented in Reference 2, was on the flex straps and bolted electrical joints, and not the lead extensions and connectors. While the entire TF bundle/flex joint/extension assembly is included in the finite element model, certain portions of the model are more finely detailed than others. Since the lead extensions were not the focus of the analysis at that time, they were rendered rather coarsely. Plus the design of the joints, connector 'A', in particular, has changed. Nevertheless, stresses can be extracted from the results of these analyses and provide valuable insight into the adequacy of the most up-to-date design of the lead extension connectors.

All of the analysis models, results and databases were saved and stored prior Willard's departure. In response to the recently updated design, coupled with the fact that the stresses in the lead extensions and connectors were never extracted, these models were re-loaded by A. Brooks and the results were examined for the first time.

Detailed examination of the model and analysis confirms that it represents scenario #82 with only electromechanical and thermal loading. There are no plasma effects. Refer to Appendix C which contains email exchanges detailing the loading checks performed.

This analysis is based on use of T. Willard's global model of the flex and connector assembly. In order to study the local stress concentrations in the partial weld and corner stress concentrations, sub models of the connectors were used. The differences between the final design and the original Tom Willard qualified design are sufficient that if another analysis of the flag connectors is undertaken, a new global model, with the proper geometry updates, is required. This would allow the proper superposition of thermal and Lorentz loads and would allow a better assessment of the loading R value which is an important contributor to the fatigue evaluation.



Figure 7-1 – NSTX-U TF Bundle/Flex Joint/Extension Assembly Model (Extensions & Connectors in 'green')



Figure 7-2 – NSTX-U TF Bundle/Flex Joint/Extension Connectors

8.0 Analysis Results

Figure 8.0-1 shows an overview of the stress intensity in connectors 'A', 'B' and 'C' for load scenario # 82. For the purposes of this review, the stresses in the holes (where the peak values are located), will be ignored. Rather, the focus will be to examine the stresses along the electron beam weld (EBW) lines.



Figure 8.0-1 – Overview of Stress Intensity Values (psi) in Connectors 'A', 'B' and 'C' Prior to Weld Updates

8.1 Type 'A' Conductor

Figure 8.1-1 is the drawing of the type "A" conductor before the e-beam weld root clean-up was applied. Figure 8.1-2 is a more detailed view of the stresses in connector 'A'. Note that in the current, updated design, the plate portion with the nine hole pattern is now EBW'd to the plate with four holes in a flush, continuous manner (see drawing E-DC1456). The fillet with the sharp corner edge is no longer present. It is just squared off. In addition, the plates are EBW'd edge to edge with no plate notch in the four-holed plate to accommodate the nine-holed plate. Figure 8.1-3 is an extreme close up of the region (cut-away view) where the plates are to be joined. The finite element mesh has been overlayed for reference. Note the coarseness of the mesh. Also note the discontinuity of the meshes of the two plates where they meet. Although the two plates are joined and the model continuous, the tying of two coarse meshes in this region will have a tendency to be overly conservative when extrapolating stresses in corners. Notice how the average stress through the middle of the joined region varies from about 6-10 ksi (light blue into green). However, the last element in the fillet corner rapidly changes five contour colors from green to red. This large stress gradient in one element is generally considered unacceptable as a finite element result and an indication that a finer mesh is needed. Accordingly, it is felt that the peak stress is probably closer to 12 ksi in this region. That includes the fillet plate effect which is also a conservative feature in the model of connector 'A'. The 12 ksi result is well within the allowable stress for EBW'd CuCrZr, so the analysis shows that the joint design is acceptable.



Figure 8.1-1 Type "A" Conductor



Figure 8.1-2 – Stress Intensity Values (psi) in Connector 'A'



Figure 8.1-3- Cutaway View of Stresses (psi) in Connector 'A' Joint

8.2 Type 'B' Conductor

A drawing of the type B connector in included in Appendix B. Figures 8.2-1 and 8.2-2 are a wide view and cutaway close-up view, respectively, of the joint region of connector 'B'. As was noted for connector 'A', the meshing is quite coarse and discontinuous across the joint.



Figure 8.2-1 Stress Intensity Values (psi) in Connector 'B'



Figure 8.2-2 Cutaway View of Stresses (psi) in Connector 'B' Joint

One element has a large stress gradient of five contour colors which likely indicate an over conservative stress result. The peak stress is not likely to exceed 12 ksi, which is well within the allowable stress for the EBW'd CuCrZr joint.

8.3 Type 'C' Conductor



Figure 8.3-1 Drawing of the Type C connector, Before Relief of Stress Concentrations

8.3.1 Stresses in the Type "C" Conductor

Figures 8.3.1-1 and 8.3.1-2 are a wide view and cutaway close-up view, respectively, of the joint region of connector 'C'. As was noted for the other connectors, the meshing is quite coarse. The edge elements have a large stress gradient of five contour colors which, once again is a likely indicator of an over conservative stress result. The peak stress is not likely to exceed 20 ksi, which is within the allowable stress for the EBW'd CuCrZr joint. Based on the results of Tom Willard's model, connector 'C' is to be considered as the most highly stressed connector of the three examined.



Figure 8.3.1-1 Stress Intensity Values (psi) in Connector 'C'



Figure 8.3.1-2 - Cutaway View of Stresses (psi) in Part of the Connector 'C' Joint



Figure 8.3.1-3 – T. Willard's Model of the Type "C" Conductor and the e beam weld detail



Figure 8.3.1-4 – Sub Model of the Type "C" Conductor Modeling the .62 in Thin Section



Figure 8.3.1-5 – Sub Model of the Type "C" Conductor Modeling the .62 in Thin Section



Figure 8.3.1-6 – Type "C" Weld Relief – As Machined

With large radius gone, Stress Concentrations appear, but are in the region of model that was extended to fit between Flex and Outer Leg in existing model



Figure 8.3.1-7 – Type "C" Design and Analysis Progression

With a sharp corner modeled at the lower end of the flag weld, the stress is 42 ksi (above, Figure 8.3.1-7). If an undercut with a radius of .25 is used, the connector at the e-beam weld, the Tresca stress drops from 42 ksi to just under 30 ksi. This is the result that will be used in subsequent fatigue calculations.

8.3.2 Cyclic Loading and R Value

The R value was estimated from the vertical field values for the 96 equilibria listed in the design point spreadsheet. Art Brooks has pointed out that when you add the thermal and radial toroidal field effects, the R value may be higher, but the estimate outlined below should be conservative.



8.3.3 S-N Fatigue and Usage Factor Calculation

The first estimate of the thinning needed to clean off the back side of the partial penetration e-beam weld was that a.72 inch thick section would remain. With a 79 ksi ultimate stress for the CuCrZr, then all the flags would satisfy normal fatigue allowables - even with the flag thinned to .72 inches at the partial EB weld.



Figure 8.3.3-1 SN Evaluation Based on 30 ksi Peak Tresca Stress and a 79 ksi Tensile Strength

The results in figure 8.3.3-1 are acceptable for the 20,000 cycle life requirement, but the measured tensile strength of the CuCrZr as delivered was not 79 ksi, but 65 ksi. (See Appendix D). Re-doing the analysis with the S-N curve scaled down by the ratio of the ultimate strengths, and taking credit for the spectrum of TF loading in the first year of operation only produced a 2000 cycle allowed life.



Figure 8.3.3-1 SN Evaluation Based on 30 ksi Peak Tresca Stress and a 65 ksi Tensile Strength

The SN data in Figure 8.3.3-1 is scaled down by the ratio of the ultimate stress from the mill certs in Appendix D and that shown in the SN plot in section 6.0. –a factor of 65/79. The result was only a 2000 full power shot cycle life. Stefan Gerhard was asked what the actual TF shot spectrum would look like in the first year. He provided the following data[5]:

- <u>16 run weeks, 5 days/week, 8 hours/day, 3 shots/hour = ~2000 shots (1920 in reality)</u>
- We stated in the FWPs that we would go at high as 0.8 T, at least on occasion
- We will commission operations at 0.55 tesla.
- Operation beneath 0.45 T will be very limited.
- CHI and RF will want the highest TF that they are allowed to use.
- Shot spectrum to assume:
 - 5% of shots at 0.45 T = 100 shots
 - 30% of shots at 0.55 T = 600 shots
 - <u>25% of shots at 0.60 T = 500 shots</u>
 - <u>25% of shots at 0.70 T = 500 shots</u>
 - 15% of shots at 0.80 T = 300 shots

The usage factor calculation was implemented in a spreadsheet and the usage factor for the first year was well below 1.0



Figure 8.3.3-1 SN Usage Factor Based on 30 ksi Peak Tresca Stress and a 65 ksi Tensile Strength, and The First Year Shot Spectrum

The problem connector and the predicted peak stress area are inspectable, so, it would be appropriate to add it to the inspection list and accept a 65 ksi ultimate temper of the CuCrZr.

Appendix A

Excerpts from T. Willard peer review presentation of TF lead extensions and support bracket proposed designs (presented 2/14/13)

NSTX Upgrade TF Lead Extensions Support Bracket Proposed Designs

02-04-13



NSTX Upgrade TF Lead Extensions Support Brackets Proposed Designs







Material: A286, HT/Aged per AMS 5525





NSTX Upgrade TF Lead Extensions SupportBrackets ProposedDesigns Cross Section View



Center TF Lead Extension Support Bracket w/ FR4 Insulator Design Shown with Support Brackets 2 & 3 Removed



Proposed TF Lead Extension Support Brackets: Bolted Joint Detail Cross Section View 1



Center TF Lead Extension Support Bracket: Bolted Joint Detail Cross Section View 2



Center TF Lead Extension: Outer Flag Helicoil Mtg Holes



TF Lead Extension SupportBrackets: 360° Circular Pattern View Showing Adequate Toroidal Clearance for Bolt Installation on Bottom Joints



Baseline-Design TF Outer Leg Lead Extensions: 1.00" CuCrZr Plates Water Cooling Maximum Thru Hole Diameter = .25"

Appendix B











Appendix C

Email correspondence

2/21/2014

PPPL Mail - Check on flex model loads



Irving Zatz <zatz@pppl.gov>

Thu, Jan 23, 2014 at 5:01 PM

Check on flex model loads

2 messages

Arthur Brooks <abrooks@pppl.gov> To: Irving Zatz <zatz@pppl.gov> Cc: Peter Titus <ptitus@pppl.gov>

Irv,

I looked at the reaction loads on the flex model to see what loads where applied. The net reactions were Fx=11827 lbs, Fy=-71635 lbs, Fz=14310 lbs with x radial, y vertical and z OOP. This agrees with what I get doing an analytic check of the TF vertical force with Btf=1 T at R=.9344 as designed. The OOP load agrees roughly with the ~.2 T vertical field from scenario #82 at the flex crossed with the 130 kA current per turn.

There doesn't seem to be any other structural load case result in the rest of the WB files, just the thermal and EM results.

So I think you've got it all.

Art

Irving Zatz <zatz@pppl.gov> To: Arthur Brooks <abrooks@pppl.gov> Cc: Peter Titus <ptitus@pppl.gov>

Art:

That sounds good to me and is consistent with Willard's report (except for writing scenario #81 in his summary).

Thanks again for checking these numbers.

Irving [Quoted text hidden]

https://mail.google.com/mail/u/D/?ul=2&/k=a1c27e91a0&v/ew=pt&search=inbcx&/h=143c11fb/9fdd1e3

1/1

Thu, Jan 23, 2014 at 5:04 PM

PPPL Mail - Re: Code for PF Field Calculations

2/21/2014

() PPPL

Irving Zatz <zatz@pppl.gov>

Re: Code for PF Field Calculations

1 message

Peter Titus <ptitus@pppl.gov> To: Arthur Brooks <abrooks@pppl.gov> Cc: Irving Zatz <zatz@pppl.gov> Fri, Jan 24, 2014 at 12:28 PM

Thanks . As a minimum can Irv add the emails to the calculation to document the check of Tom's input, and the check of the post disruption currents? I think the max post disruption vertical field of .25 T vs .23 T which was the basis of the loads Tom used, is not going to violate the margins that Tom had for the strap. -Peter

On Fri, Jan 24, 2014 at 11:53 AM, Arthur Brooks <abrooks@pppl.gov> wrote:

Peter,

Tom's model does NOT contain the plasma, just the OH, PF1-5 and the TF, which is good. The current values do agree with the scenario #82 values. The vertical fields at the Flex Strap are higher for this scenario without plasma 0.23 T vs 0.19 T with plasma.

The Post Disruptions currents give rise to slightly higher fields and occur at scenario #31 .Without plasma the vertical field is 0.25 T vs 0.21 T with plasma.

Art

On Thu, Jan 23, 2014 at 3:55 PM, Peter Titus <ptitus@pppl.gov> wrote:

I suppose we don't know whether Tom used the with or without plasma equilibria. It would be nice to document that Toms calculation enveloped all the equilibria with and without the plasma. - and post disruption. Attached is a spreadsheet that calculates the torsional shear for Charlies post disruption data. I believe it is on the web in his design point spreadsheet, or you can trust that I extracted it properly. Either way could you run it through your code and see if it is substantially higher than the nominal fields? Can you find the currents input to Maxwell to determine if Tom used the "without plasma" data? -Peter

On Thu, Jan 23, 2014 at 3:15 PM, Arthur Brooks <abrooks@pppl.gov> wrote: Peter,

For now I have a code on the Unix Cluster in directory:

/p/eaddata/abrooks/nstx_csu/pfcalc

The code executable is called pfcalc3.x; the FORTAN source is pfcalc3.f. It reads the scenario data from the file pfcalc3.inp and point data from the file pfcalc3.pts (you shouldn't need to change the scenario data). It outputs the max values in pfcalc3.max and all values in pfcalc3.brbz.

For the TF Flex, the max vertical field magnitude at r=0.487, z=2.849 is for scenario 82 and is 0.19T with plasma. Without plasma it increases to 0.23T (also for scenario 82).

The full set of fields is plotted in the attached spreadsheet for the TF Flex location.

https://mail.google.com/mail/u/0/?ul=2&/k=a1c27e91a0&vlew=pt&search=inbox&th=143c5527b675631c

Appendix D Mill Certs for the CuCrZr Plate



60 RADO DRIVE NAUGATUCK, CT 06770 203-729-1111 FAX: 203-729-1919

RESISTANCE WELDING ELECTRODES AND ACCESSORIES COPPER ALLOY RAW MATERIALS

MATERIAL CERTIFICATION

Customer: Martinez & Turek, Inc.

Order #: 1402860

Customer P.O. #: 69416

| Dimension: | 1" x 3" x 8.120" C2 C18150 | (26PCS) | Qty.: 230 lbs. |
|------------|---------------------------------|---------|----------------|
| | 1" x 4.620" x 5" C2 C18150 | (26PCS) | Qty.: 217 lbs |
| | 1" x 3.250" x 7.120" C2 C18150 | (14PCS) | Qty.: 118 lbs |
| | 1" x 4.500" x 14.500" C2 C18150 | (14PCS) | Qty.: 325 lbs |
| | 1" x 2.620" x 7.120" C2 C18150 | (14PCS) | Qty.: 96 lbs |
| | 1" x 2.500" x 7.120" C2 C18150 | (14PCS) | Qty.: 92 lbs |
| | 1" x 4.120" x 14.500" C2 C18150 | (14PCS) | Qty.: 298 lbs |
| | 1" x 3.620" x 7.120" C2 C18150 | (14PCS) | Qty.: 131 lbs |
| | 1" x 3.250" x 7.120" C2 C18150 | (14PCS) | Qty.: 118 lbs |
| | 1" x 4.562" x 14.500" C2 C18150 | (14PCS) | Qty.: 325 lbs |
| | 1" x 1" x R/L'S C2 C18150 | (4PCS) | Qty.: 230 lbs |

Heat No: 32184

Mechanical/Physical Test Results:

Conductivity: 84/87 % I.A.C.S. Hardness Rockwell "B": 72/77 $^{R}_{b}$ Tensile: 65,500 PSI Yield Strength: 52,000 PSI Elongation in 2 inches or 4 D %: 29 % (2 ")

Chemical Analysis:

Chromium: 0.690 % Zirconium: 0.090 %

Copper including Silver plus named elements 99.70 % Min:

Balance

WE HEREBY CERTIFY that the material shipped on the above order has been produced in accordance with the above specification/order by our standard practice. We further certify that no Mercury was in contact with the metal at any time during its manufacture and testing.

Norman L. Finke gr.

2014 Norman L. Finke Jr. Quality Assurance Manager

October 6.

Date Approved

ISO DOC.FORMS\F-4.2.4-6 Rev A 12/02