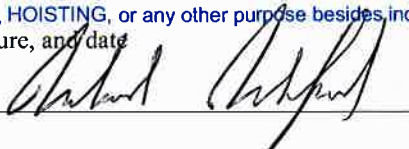



PPPL	PRINCETON PLASMA PHYSICS LABORATORY	PROCEDURE	No. ENG-033 Rev 5 Attachment 1
	PPPL Calculation Form		Page 1 of 1

TCR-ENG-033,R5-002

PPPL Calculation Form			
Calculation #	<u>14161003_MM 2016-10-03</u>	Revision #	<u>0</u>
		WP #, if any (ENG-032)	<u>2213</u>
Purpose of Calculation: (Define why the calculation is being performed.)			
This calculation qualifies the use of a "friction clamp" to grip a copper bar and apply load during testing of winding equipment during setup for the PF1A coil winding.			
References (List any source of design information including computer program titles and revision levels.)			
None			
Assumptions (Identify all assumptions made as part of this calculation.)			
See Attached			
Calculation (Calculation is either documented here or attached)			
See Attached			
Conclusion (Specify whether or not the purpose of the calculation was accomplished.)			
<ol style="list-style-type: none"> The clamp is qualified for 1400 lbf. QTY 8x 3/8-16 UNC bolts shall be torqued to 15 ft-lbfs before use. Faying surfaces of the clamp faces shall not touch in order to ensure the load path is through the copper Swivel Hoist rings shall be used per SOP. (Depth of thread, Torque, Contact Area, etc.) Rigging shall be arranged such that the direction of pull is straight (i.e. no bending moments induced in copper). THIS PART IS NOT TO BE USED FOR LIFTING, HOISTING, or any other purpose besides indicated 			
Cognizant Engineer (or designee) printed name, signature, and date			
MICHAEL MAROENFELD  10/31/16			
I have reviewed this calculation and, to my professional satisfaction, it is properly performed and correct.			
Checker's printed name, signature, and date			
MARC J. SIBILIA  10/3/16			

PPPL	PRINCETON PLASMA PHYSICS LABORATORY	PROCEDURE	No. ENG-033 Rev 5 Attachment 2
Minimum Requirements for Checking of Calculations			Page 1 of 1

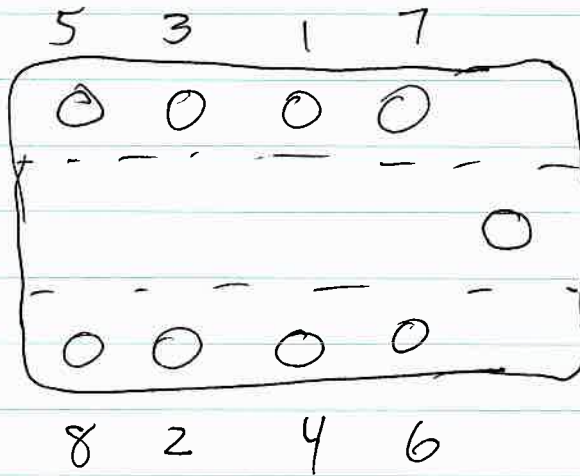
TCR-ENG-033,R5-002

1. Assure that inputs were correctly selected and incorporated into the design.
2. Calculation considers, as appropriate:
 - Performance Requirements (capacity, rating, system output)
 - Design Conditions (pressure, temperature, voltage, etc.)
 - Load Conditions (seismic, wind, thermal, dynamic)
 - Environmental Conditions (radiation zone, hazardous material, etc.)
 - Material Requirements
 - Structural Requirements (foundations, pipe supports, etc.)
 - Hydraulic Requirements (NPSH, pressure drops, etc.)
 - Chemistry Requirements
 - Electrical Requirements (power source, volts, raceway, and insulation)
 - Equipment Reliability (FMEA)
 - Failure Effects on Surrounding Equipment
 - Tolerance Buildup
3. Assumptions necessary to perform the design activity are adequately described and reasonable.
4. An appropriate calculation method was used.
5. The results are reasonable compared to the inputs.

NOTE: IT IS THE RESPONSIBILITY OF THE CHECKER TO USE METHODS THAT WILL SUBSTANTIATE TO HIS/HER PROFESSIONAL SATISFACTION THAT THE CALCULATION IS CORRECT.

BY SIGNING CALCULATION, CHECKER ACKNOWLEDGES THAT THE CALCULATION HAS BEEN APPROPRIATELY CHECKED AND THAT THE APPLICABLE ITEMS LISTED ABOVE HAVE BEEN INCLUDED AS PART OF THE CHECK.

Manu Stiles
10/3/16



Bring nuts to finger tight in sequence
 Tighten Torque in sequence to 'snug'
 Torque in sequence to 5 ft lb.
 Torque in sequence to 15 ft lb.
 Re torque " " to 15 ft lb.

Torque nuts; not bolts.

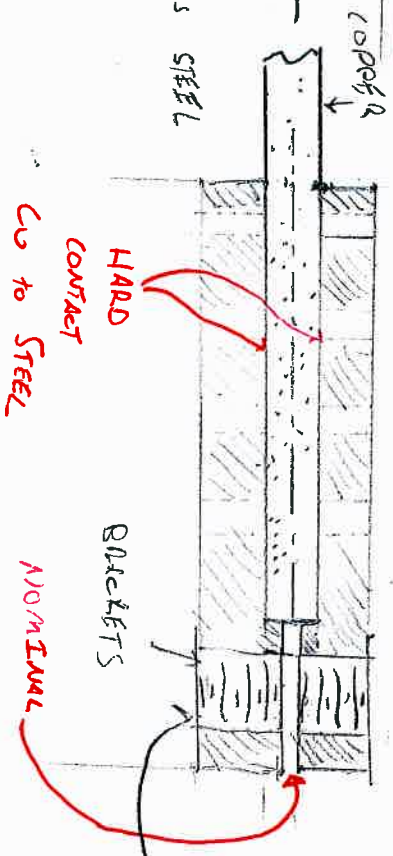
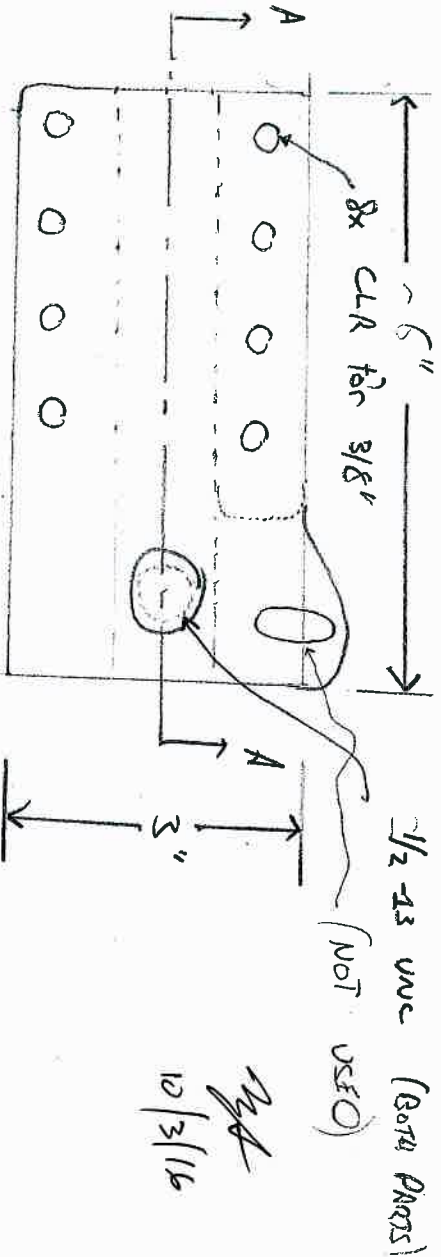


Top spot
OF COPPER

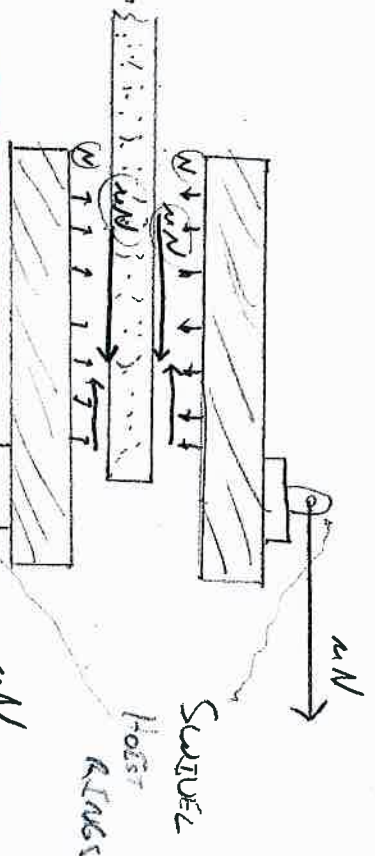
- * BRACKET APPEARS TO BE STAINLESS STEEL
- * OF THE COPPER COATED WITH POLYIMIDE FORMER

- * COPPER CROSS SECTION $\sim 1/2" \times 1"$
- * SLOT HAS NOMINALLY $\sim 0.10"$ CLEARANCE
- * HORIZONTAL ACROSS COPPER

- * STEEL FINISH SURFACES
- * HAVE A NOMINAL CLEARANCE OF $\sim 0.10"$
- * CAP MUST BE PRESERVED TO ENSURE



- * TAP DEPTH FOR SQUARE HOIST RING $\sim .7" \rightarrow$ MEET CROSS HATCH REQUIREMENTS
- * NOTICE FINGERING FORCE IS APPLIED TO BOTH TOP & BOTTOM OF COPPER



- * RIG TOP & BOTTOM TO AVOID REMOVING MOMENT

SAFETY FACTORS

* 1.5 x for STATIC LOAD

* 2x for coefficient of friction

* 1.33x for bolt pretension

↳ $n_1 \cdot n_2 \cdot n_3 = n_{tot} = 4x$

Coefficient of friction, $\mu = .146$ ✓

↳ BY MEASUREMENT WITH "RAMP TEST" ✓

See actual parts

10/3/16

Assume QTY 8x 3/8-16 UNC

BOLTS ARE SOURCE TO 25 ft. lbs ✓

= 160 in. lbs

2. Bolt Torque / Pretension

$$F = \frac{T}{(0.2)/D} = \frac{180}{(1.2)(3/8)} = 2400$$

↳ 19200 lbf bolt clamp load

↳ NOTE Bolt STRESS = $\sigma = \frac{F}{A} = \frac{2400}{.0775} = \frac{191}{1} = 31 \text{ KSI}$ (ok for nominal bolts)

2. Full Torque TRACTION Q/W COPPER & STEEL AT FAILURE

$$V = 2 \mu N = (2)(.146)(19200) = 5666.4 \text{ (2)(1)(191)}$$

↳ LOAD APPLIED ON BOTH SURFACES (SEE FIG.0 SHEET 1)

3. SAFE WORKING LOAD

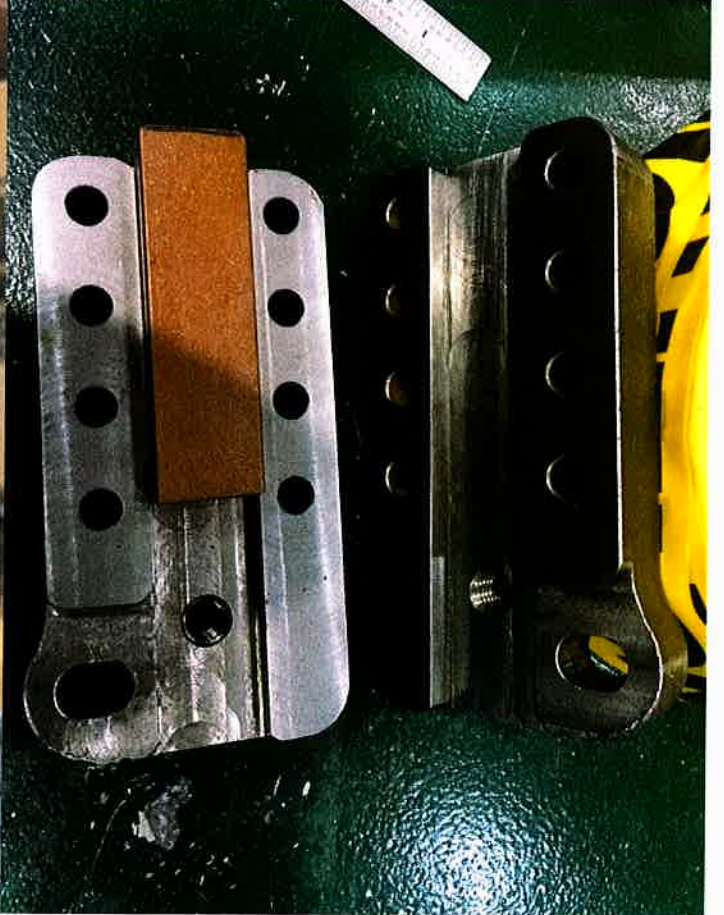
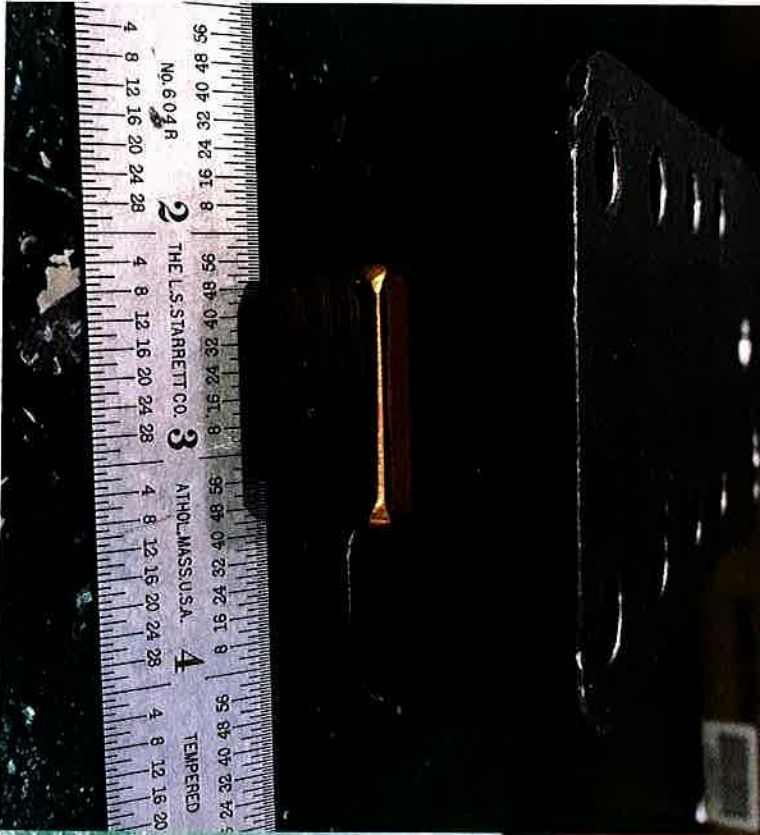
$$SWL = \frac{\text{FAILURE LOAD}}{n_{tot}} = \frac{5666.4}{4} = \boxed{1400 \text{ lbf}} = SWL \checkmark$$

4. CHECK COPPER STRESS

$$A = (1) (1.5) = .5 \text{ in}^2$$

$$\sigma = \frac{1400 \text{ lbf}}{.5 \text{ in}^2} = 2.8 \text{ KSI}$$

(LOW EVEN FOR SOFT COPPER)



2016-10-03

MARCEVITZ

AK
10/3/16